



Engine Combustion Network

ECN 4 Spray G mixing experiments and modeling

Presenters:

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IFPEN

Lyle M. Pickett

Sandia National Laboratories

Topic organizer:

Daniel Vaquerizo Sánchez

CMT – Motores Térmicos

Experiment

- Argonne National Laboratory
 - Radiography/tomography measurements: Daniel Duke, Chris Powell, Alan Kastengren, Andrew Swantek, Nick Sovis
- General Motors R&D
 - Droplet velocity/sizing: Scott Parrish
- IFPEN
 - LIF mixing: Lama Itani, Gilles Bruneaux, Laurent Hermant
- Sandia National Laboratories
 - High-speed planar imaging: Scott Skeen, Julien Manin, Lyle Pickett, Jonathan Frank, Panos Sphicas

Modeling

- Argonne National Laboratory
 - Modeling: Sibendu Som, Kaushik Saha, Z. Weng, Y. Pei, Chao Xu, (UConn student)
- Politecnico di Milano
 - Tommaso Lucchini, Tarcisio Cerri
- General Motors R&D
 - Ron Grover

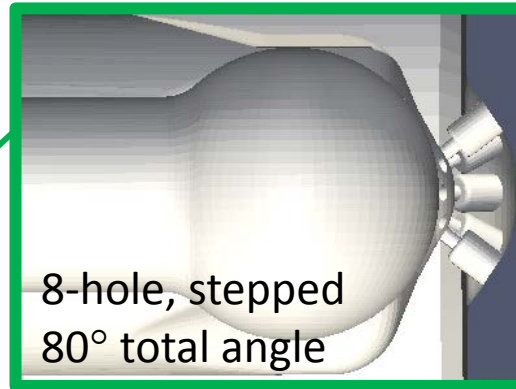


Spray G experiments and modeling have progressed...we can now ask more detailed questions about gasoline DI modeling

- Significant spray G experimental contributions



Gasoline Spray G



8-hole, stepped
80° total angle

90° C

573 K, 6 bar

Velocity & Turbulence

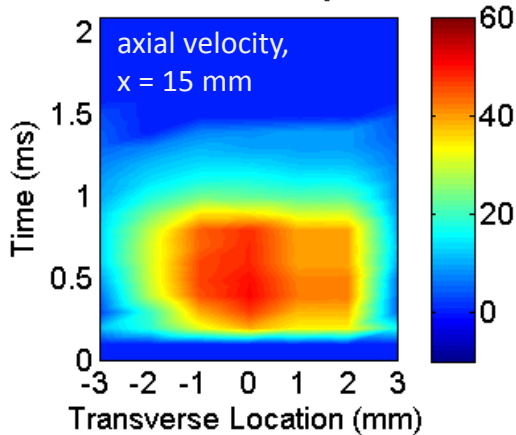
Liq. & vap. penetration
Mie/Schlieren/DBI

Liquid mixture fraction

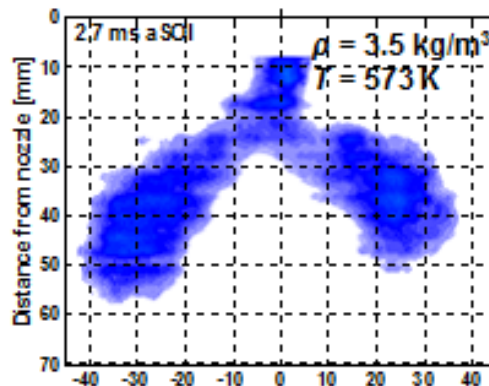
Vapor mixture fraction
in Engine

Spray structure
HS microscopy

droplet size,
velocity via PDI
GM

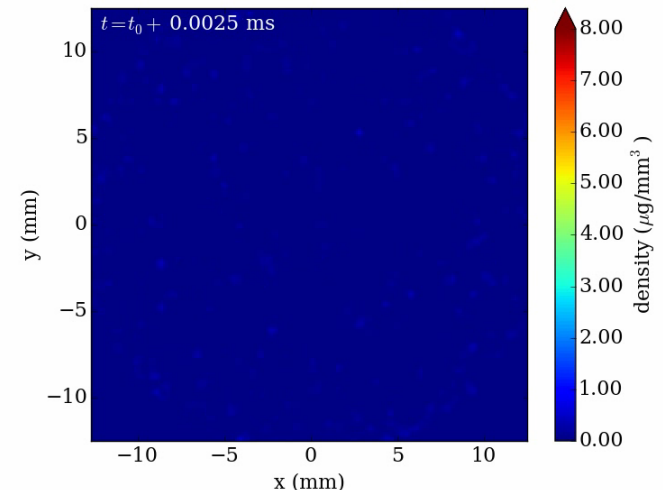


Vapor mixture
late ASI
IFPEN

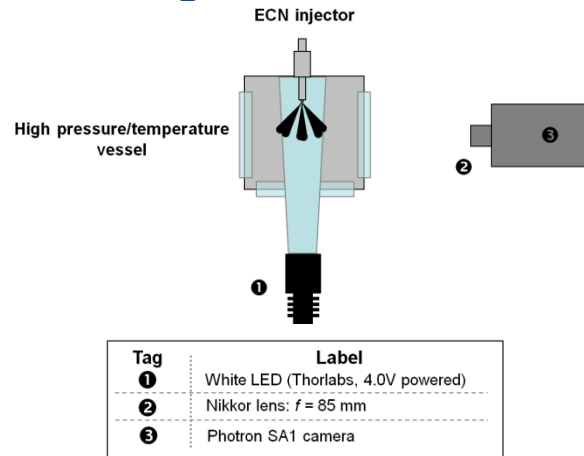


Non-vaporizing
mixing
Argonne

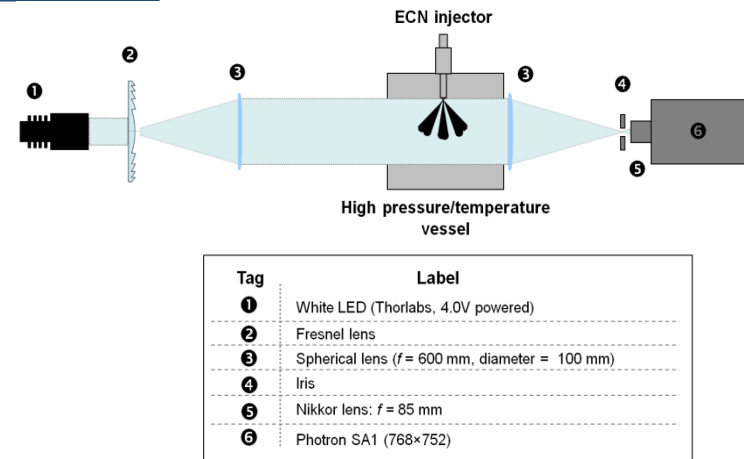
$z = 10 \text{ mm}$



Mie-scattering

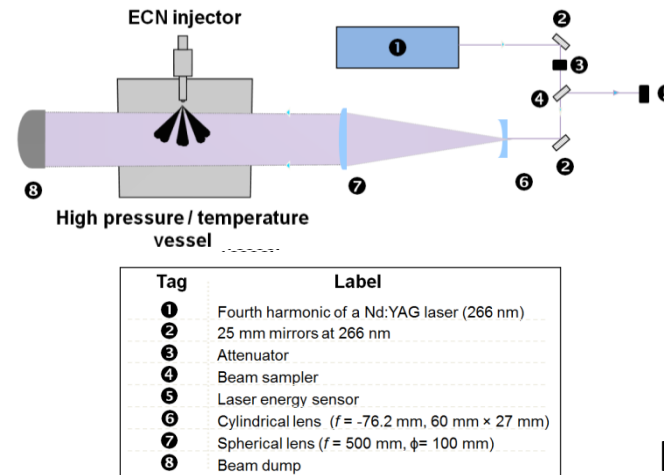


Schlieren



LIF

- Use fuel blend: iso-octane (purity grade = 99.8%) + 0.03% vol. p-DFB
- 1) two-color LIF collection, measure temperature and fuel concentration
- 2) single-color LIF collection with calibration by total injected fuel
- see SAE 2015-01-1902



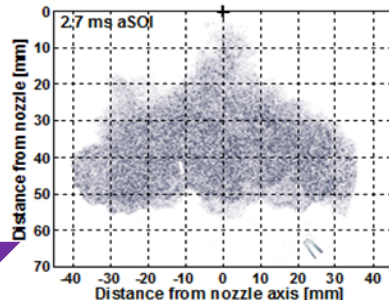
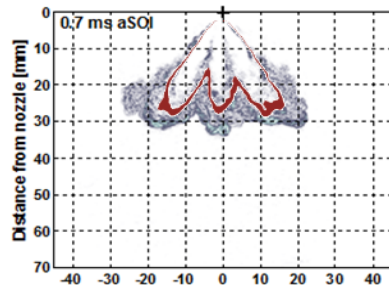
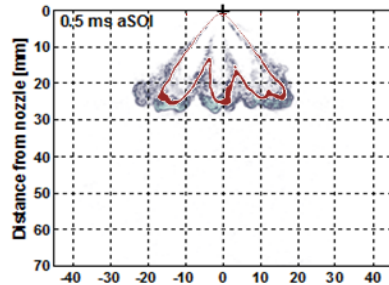
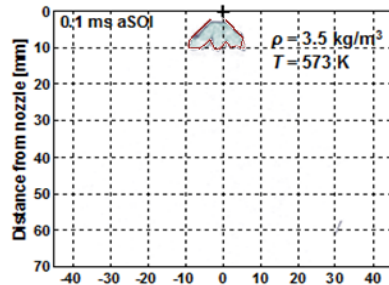
IFPEN



Background: evolution of mixing at different ambient densities

Spray G

ambient density



Results: Jet Evolution

- Four stages in the spray development were defined (0.1, 0.5, 0.7 and 2.7 ms aSOI).
- Red contours derived from instantaneous Mie-scattering images.
- Until 0.7 ms: Jet momentum is mainly driving the mixing process.
- For timings $\gg 0.7 \text{ ms}$: Jet concentrated on the injector axis where mixing is driven by aerodynamic motion created by air entrainment

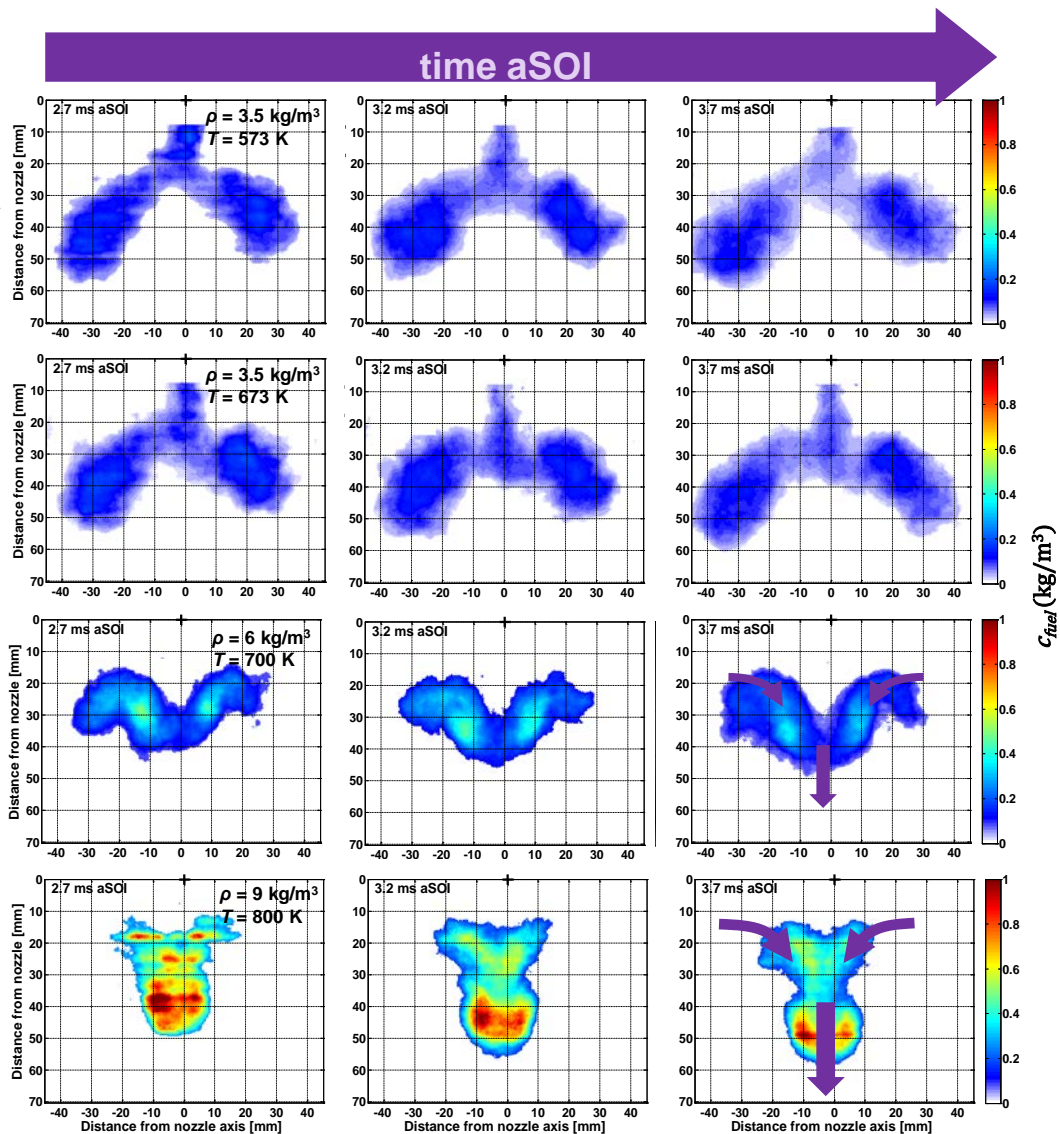
Fuel concentration with p-DFB LIF imaging

Spray G

$\rho = 3.5 \text{ kg/m}^3$
 $T = 673 \text{ K}$

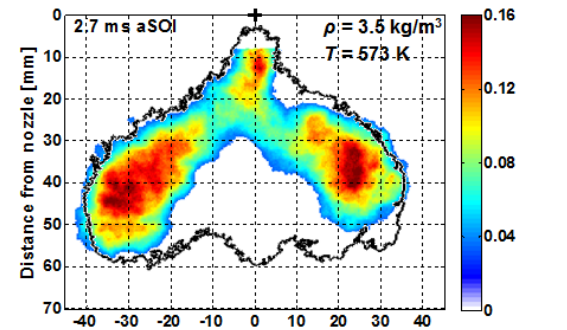
$\rho = 6 \text{ kg/m}^3$
 $T = 700 \text{ K}$

$\rho = 9 \text{ kg/m}^3$
 $T = 800 \text{ K}$

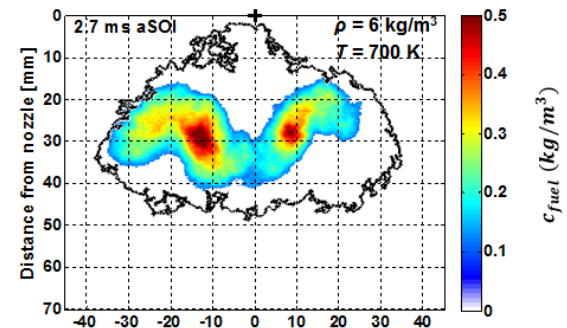


- pLIF images lie within schlieren contour
- reveal the extent of mixing at the center of the spray

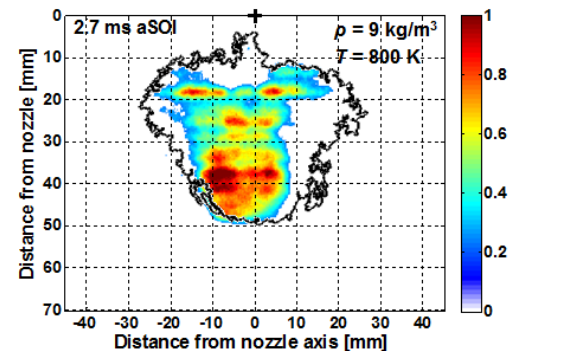
Spray G

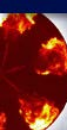


$\rho = 6 \text{ kg/m}^3$
 $T = 700 \text{ K}$



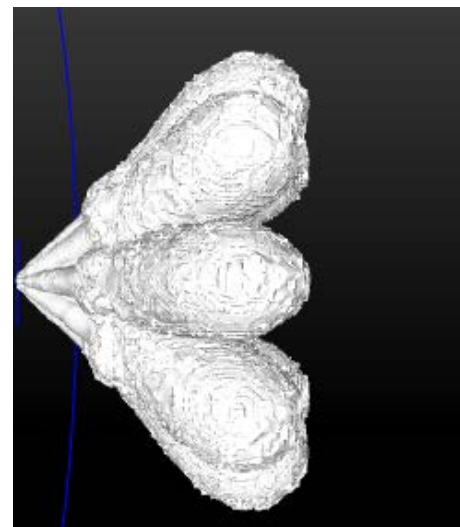
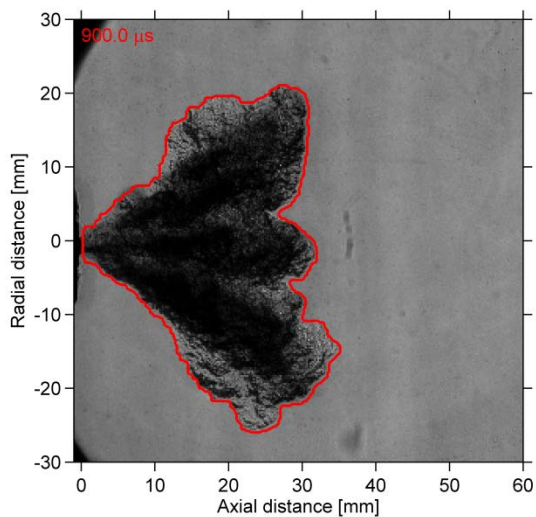
$\rho = 9 \text{ kg/m}^3$
 $T = 800 \text{ K}$





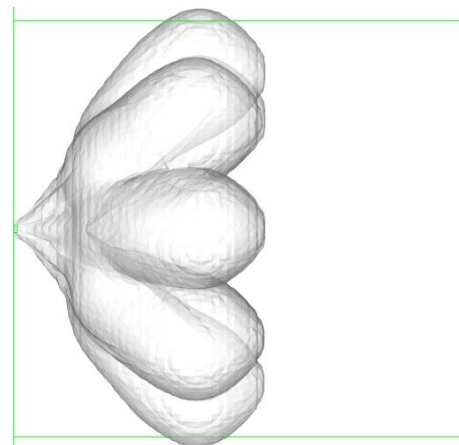
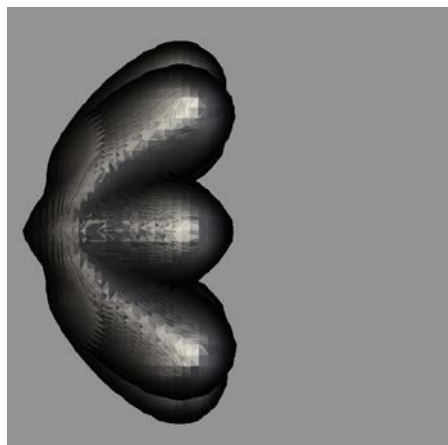
0.9 ms ASOI

Experiment



ANL

PoliMi



UW

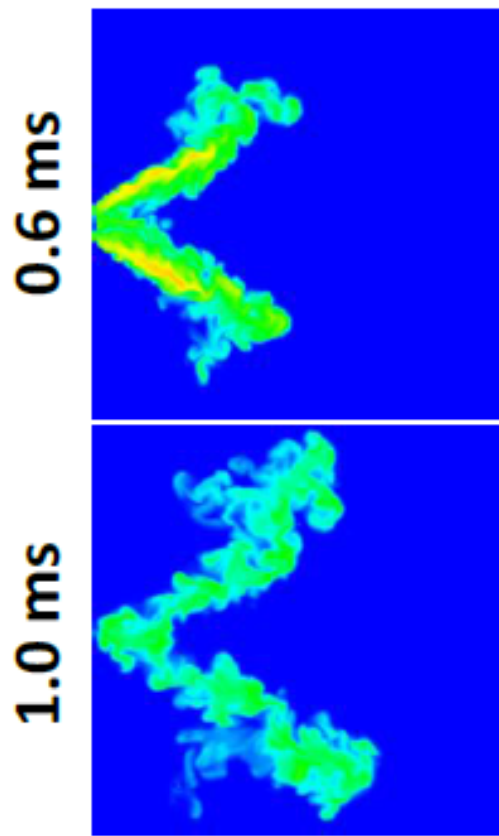


	ANL	Polimi	UW
CFD Code	Converge 2.1.0	OpenFOAM + Lib-ICE	KIVA3V-r2-ERC
Turbulence Model	Standard k- ϵ	Standard k- ϵ	RNG k- ϵ
Injection Model	Lagrangian/Blob	Lagrangian/Huh	Lagrangian/Blob
Primary Break-up Model	KH-RT (B1 7; RT Length 0.0)	Huh-Gosman	KH-RT (B1 40; RT Length 1.0)
Secondary Break-up		Wave	
Vaporization	Frössling	Spalding Number-based (mass-based)	Frössling
Heat Transfer	O'Rourke	Ranz-Marshall	Ranz-Marshall
Collision	NTC	None	
Turbulent Dispersion	O'Rourke	None	Gaussian Distribution
Droplet Drag	Dynamic	Dynamic w/ non-spherical correction	Dynamic w/ non-spherical correction

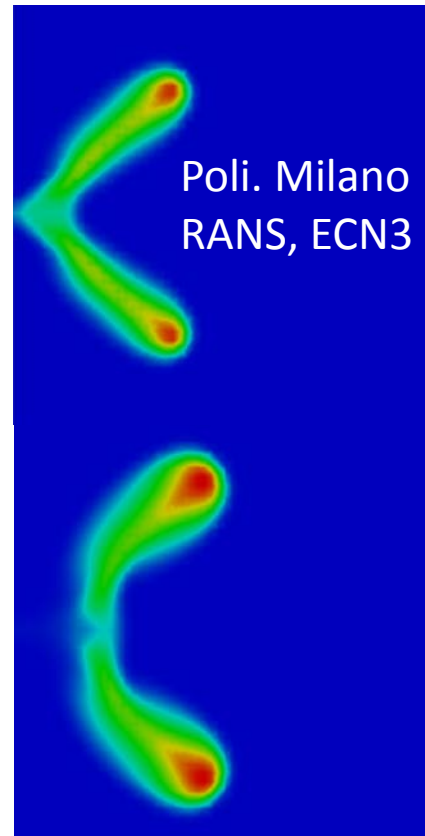


	ANL	Polimi	UW
Domain Dimensions	108 Ø x 108 mm	169 x 248 mm	100 x 100 x 60 mm
Base Cell Size	2 mm	4 mm	1 mm
Min Cell Size	0.25 mm	1 mm	-
Adaptive or Static Refinement	Both	Adaptive	Uniform
Cell Type	Cartesian	Cartesian	Cartesian Hexahedron
Total/Maximum Cell Count	1.53 Million	115,000	600,000

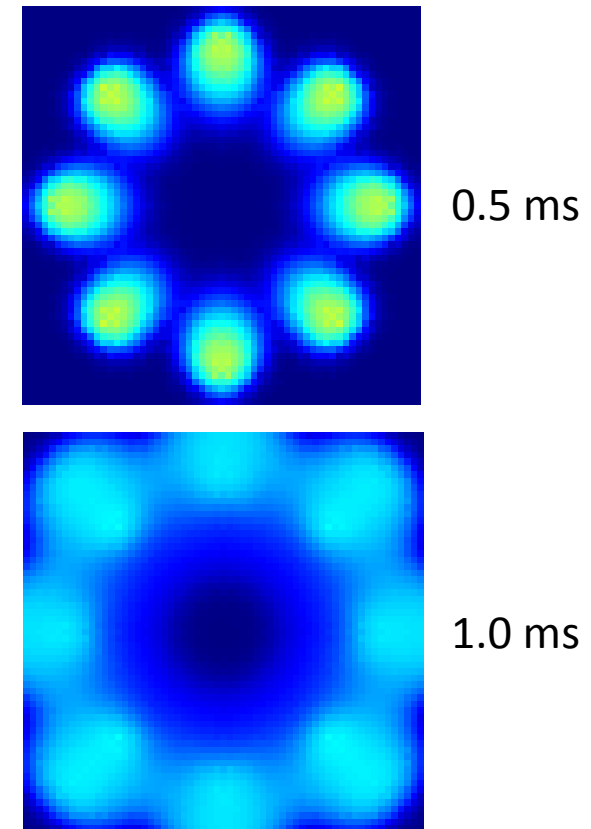
Argonne National Lab
LES
Som, Wang, Pei et al.
ASME ICE 2015-1003



Fuel mixture fraction on Y-Z plane



Poli. Milano
RANS, ECN4



Vapor fuel mixture fraction
on X-Y plane, Z = 15 mm



Simulation Set-up at ANL

Modeling tool	CONVERGE
Dimensionality and type of grid	3D, Cartesian
Grid embedding management	Base mesh size fixed to 1 mm, 0.125 mm min. cell size (a) Fixed embedding near injectors and boundaries (b) Gradient based AMR on the velocity and temp. fields
Number of processors used	Most simulations on 64 processors
Turbulence model	LES: Dynamic Structure*
Spray models	Injection: Blob injection model Breakup: KH-RT without breakup length concept Collision model: NTC (No Time Counter) Coalescence model: Post Collision outcomes Drag-law: Dynamic model
Time step	Variable based on spray, evaporation processes

Momentum Equation

$$\frac{\partial \bar{\rho} \tilde{u}_i}{\partial t} + \frac{\partial \bar{\rho} \tilde{u}_i \tilde{u}_j}{\partial x_j} = -\frac{\partial \bar{P}}{\partial x_i} + \frac{\partial \bar{\sigma}_{ij}}{\partial x_j} - \frac{\partial \tau_{ij}}{\partial x_j}$$

Dynamic Structure Model

$$\tau_{ij} = 2k\bar{\rho} \frac{L_{ij}}{L_{kk}}$$
$$L_{ij} = \widehat{\tilde{u}_i \tilde{u}_j} - \tilde{u}_i \tilde{u}_j \quad k = \frac{1}{2} (\widehat{u_i u_j} - \tilde{u}_i \tilde{u}_j)$$

Sub-grid stress tensor τ_{ij} needs to be modeled

* Dynamic structure model was chosen since it provided best predictions against diesel spray experimental data compared to the other SGS models by our team (Xue & Som et al., AAS paper, 2013)

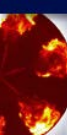
Model Settings for LES simulation

Constant	Brief Description	Value
B_0	KH model droplet size constant	0.6
B_1	KH model breakup time constant	5
C_1	KH model droplet normal velocity	0.188
C_t	RT model breakup time constant	1.0
C_{RT}	RT model droplet size constant	0.6
Shed factor	Percentage of mass given to child droplets by parent droplet	0.25
Pr_{sgs}	k-equation constant	1.39
C_ε	LES dissipation rate model constant	1.0
C_{ps}	Drop turbulent dispersion constant	0.01

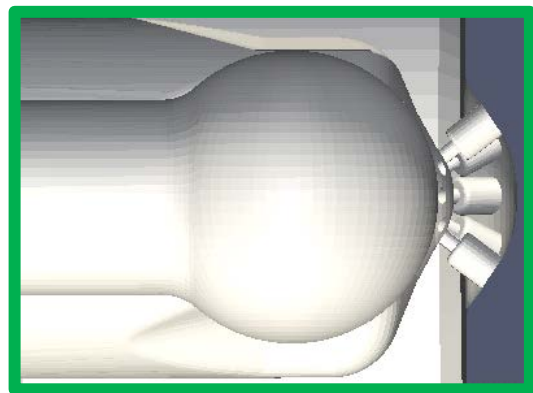
Base cell size (mm)	AMR & Embed scale	Minimum cell size (mm)	Number of injected parcels	Peak cell count at 1 ms (millions)	Wall clock time (hours) till 1 ms on 64 CPUs
1	3	0.125	50,000	~ 13.3	7.2

- 20 realizations with random number seeding (injections) performed (720 CPU hrs per injection)
 - 14,400 total CPU hrs

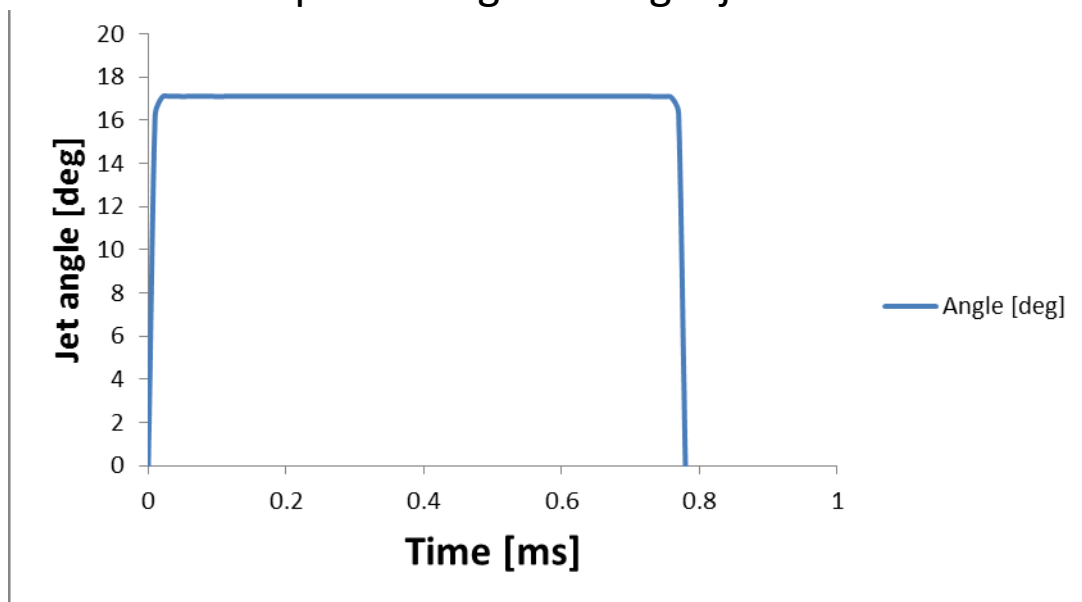
CFD Code details	
Software	OpenFOAM + LibICE libraries by Politecnico di Milano (spray, combustion, mesh management, ...)
Solver	RANS, unsteady, compressible solver. Spray modeled with lagrangian approach
Turbulence model	Standard k-epsilon with modified C1 (1.5 instead of 1.44)
Fuel type	Iso-octane
Liquid properties	Iso-octane NIST data
Gas properties	Iso-octane and N2 Janaf coefficient
Temporal Discretization scheme	Euler Implicit
Convection discretization scheme	Limited (TVD), second-order
Diffusion discretization	Including the non-orthogonal component of the gradient
Lagrangian spray model details	
Atomization Model	Huh-Gosman, tuned following LES Data presented in SAE Paper 2014-01-0149
Injection Model	Huh
Secondary Breakup model	Pilch-Erdman
Collision model	off
Evaporation model	Based on Spalding mass number
Heat transfer model	Ranz-Marshall
Turbulent dispersion model	Stochastic
Mesh details	
Cell size	Ranging from 1 to 4 mm using Adaptive Local Mesh refinement

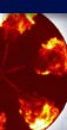


- Plume vector, or inclination angle, set at specified 37 degrees (rel. to injector), for both Argonne LES & RANS and PoliMi RANS simulations
- Plume origin at designed radius
- Plume “spreading angle” allowed to change with time based upon nozzle velocity, turbulence, etc
- Plume vector / inclination angle does not change with time



PoliMi plume angle during injection

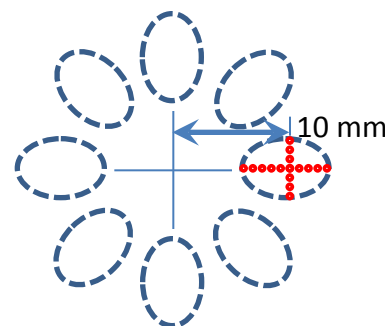
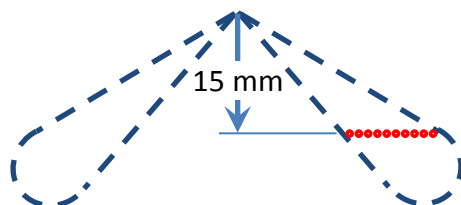




ECN

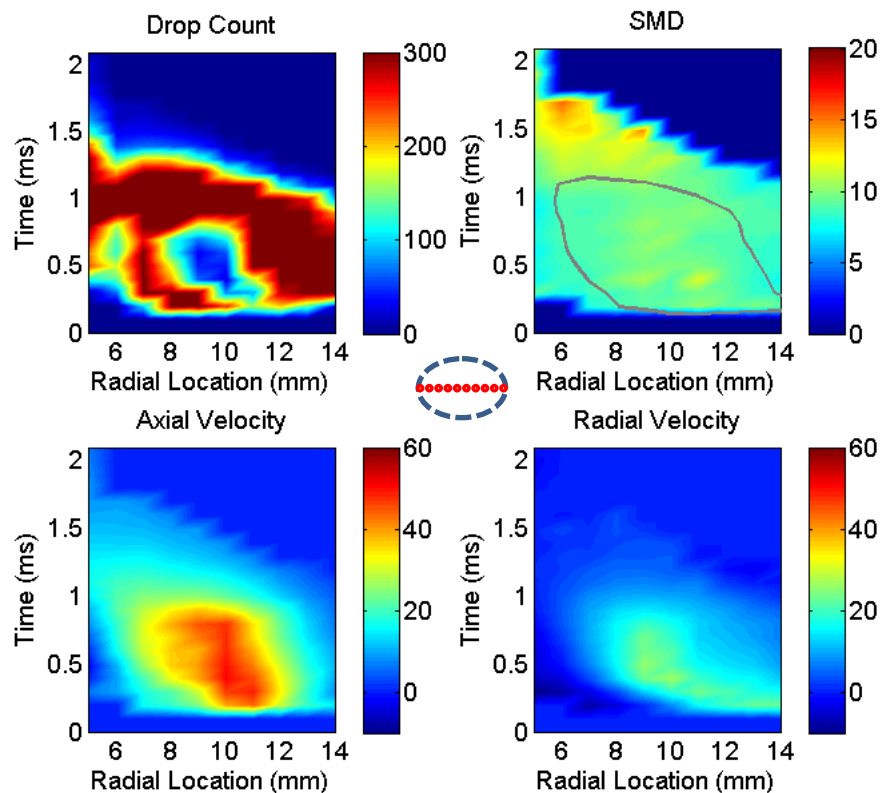
Spray G – Velocity and drop size observations

Scott Parrish, GM
ECN3

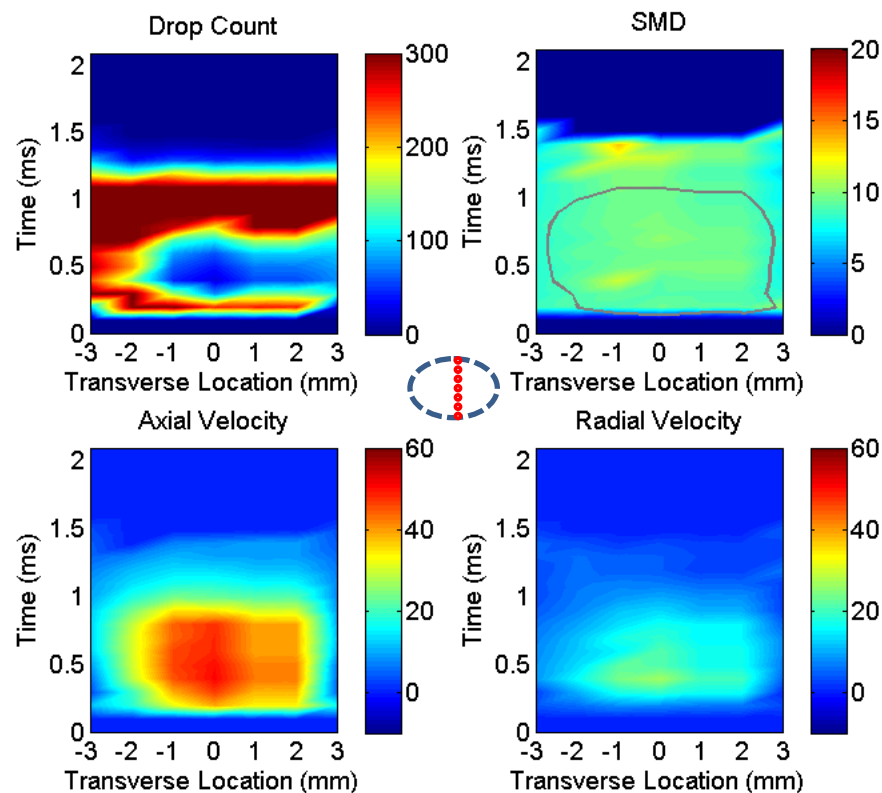


for injector #16,
plume #1:
Substantial dataset with
~ 10,000 injections

Radial Scan



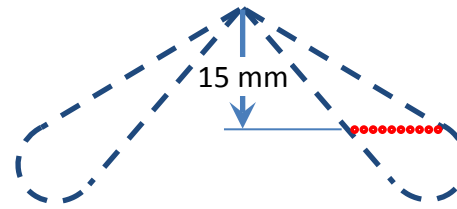
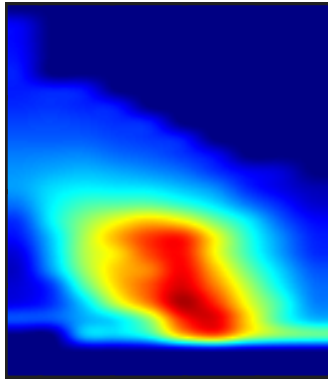
Transverse Scan



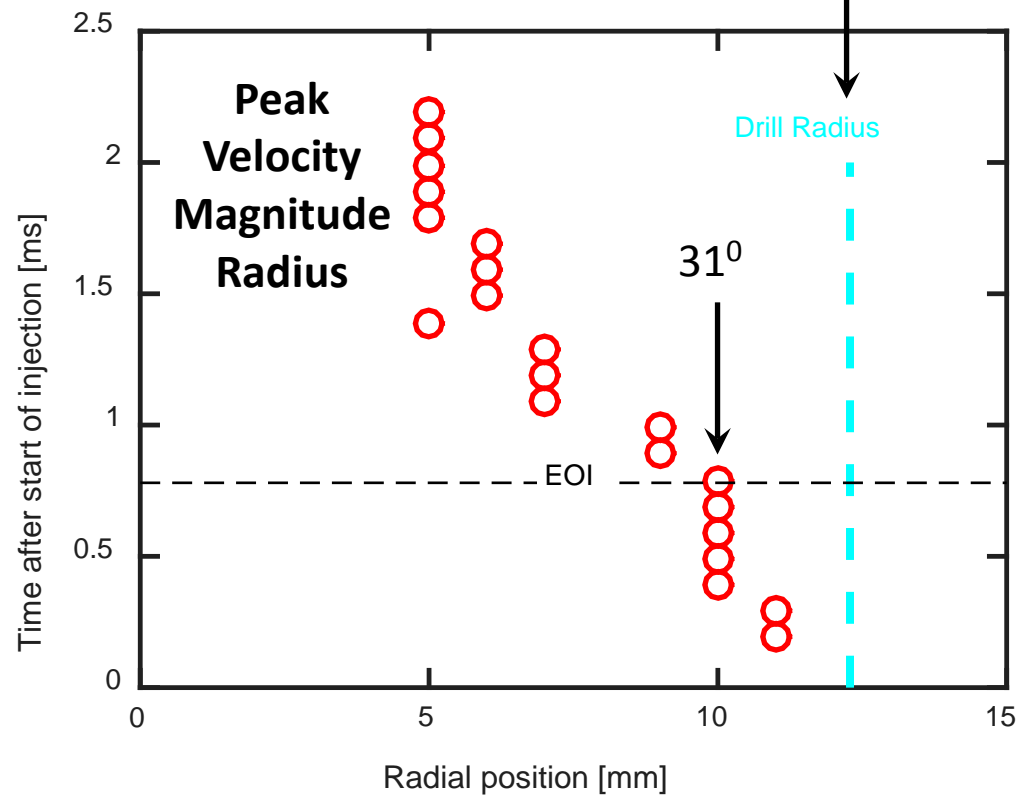
**Velocity
magnitude**

axial

radial

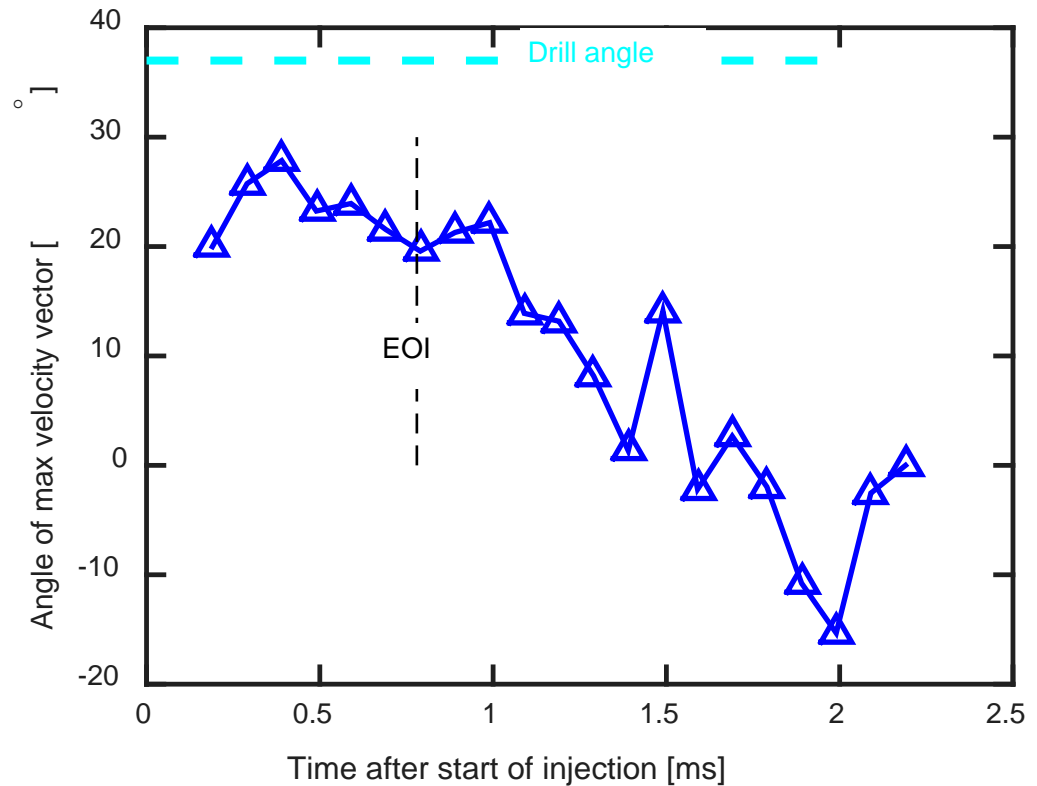
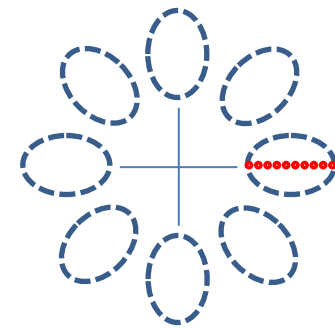
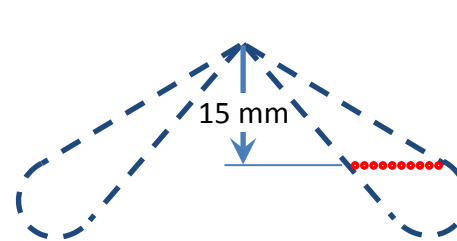
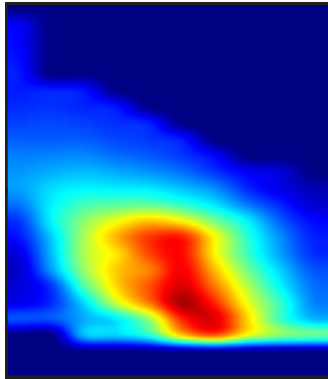
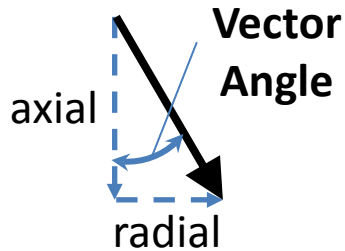


radius at 15 mm
for 37° drill
angle





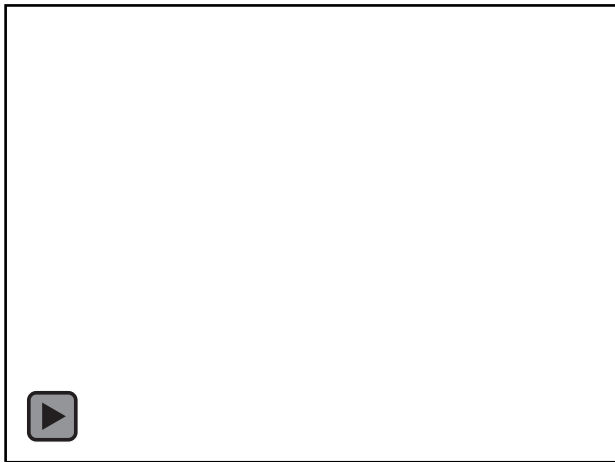
Plume inclination angle decreases with time



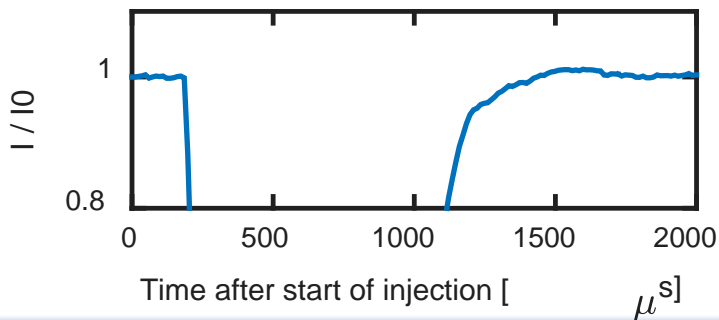


Other planar visualization shows plume redirect

Line of sight
Diffused back illumination
contrast set to $I/I_0 = 0.9$ to 1.0



Measured intensity in dark ROI



100 kHz Planar Rayleigh imaging attempt at Sandia
Standard Spray G conditions
 3.5 kg/m^3 , 573 K

lower laser energy

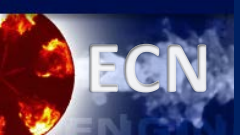


higher laser energy

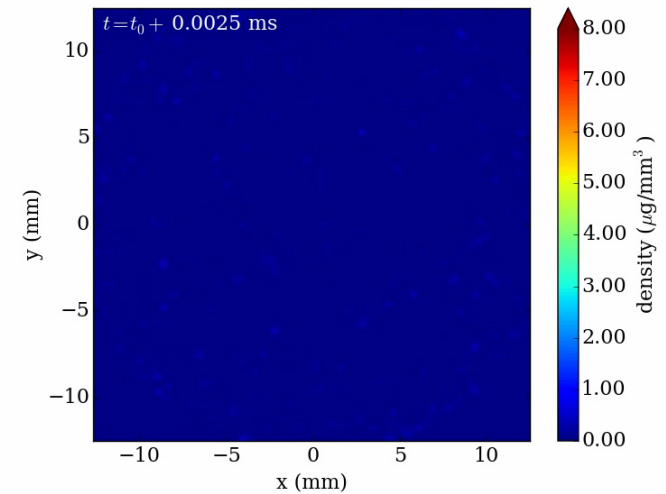
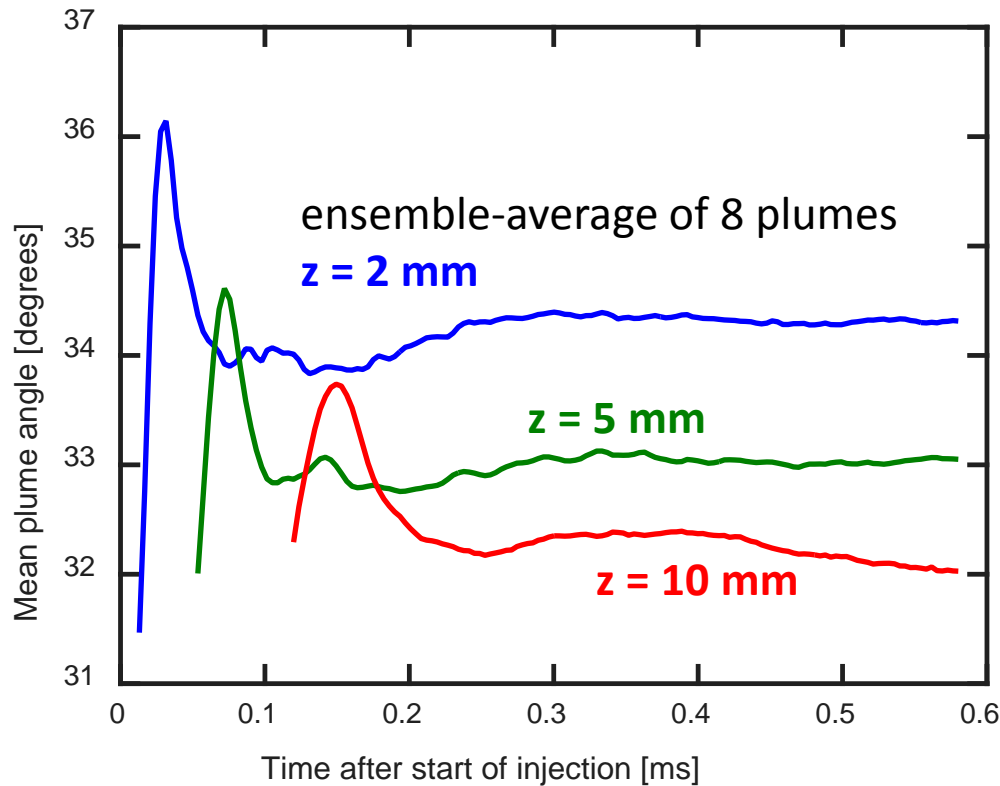


30°



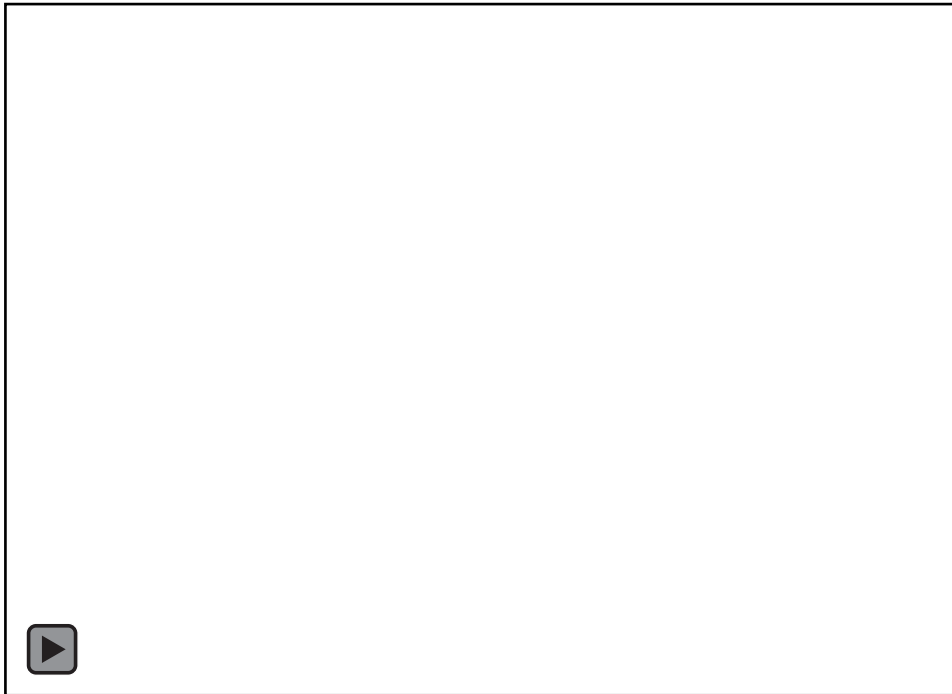


Plume pointing vector derived from radiography at different axial positions



- Gradual shift of plume direction to injector axis while moving downstream
- Marked transient period at higher angle

Schlieren imaging
9 kg/m³, 573 K



100 kHz Planar Rayleigh
Standard Spray G conditions
9 kg/m³, 1050 K

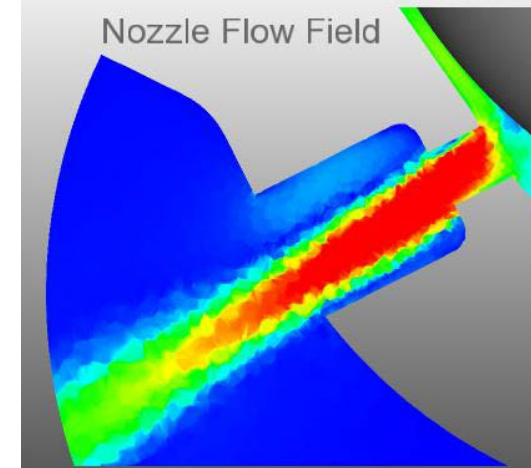


30°

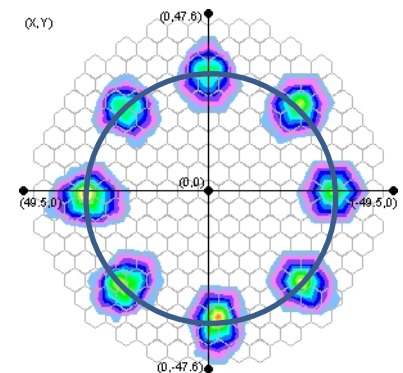


- Is the plume vector redirection caused by internal flow or downstream fluid mechanics?
- If imposing a constant plume vector (and plume spreading angle), will Lagrangian-droplet RANS and LES simulations predict plume interaction and merge?
- Lacking detailed spray measurements, how does the Lagrangian-droplet modeler predict plume vector (as a function of time)?

Spray G at ECN3
Ron Grover, GM

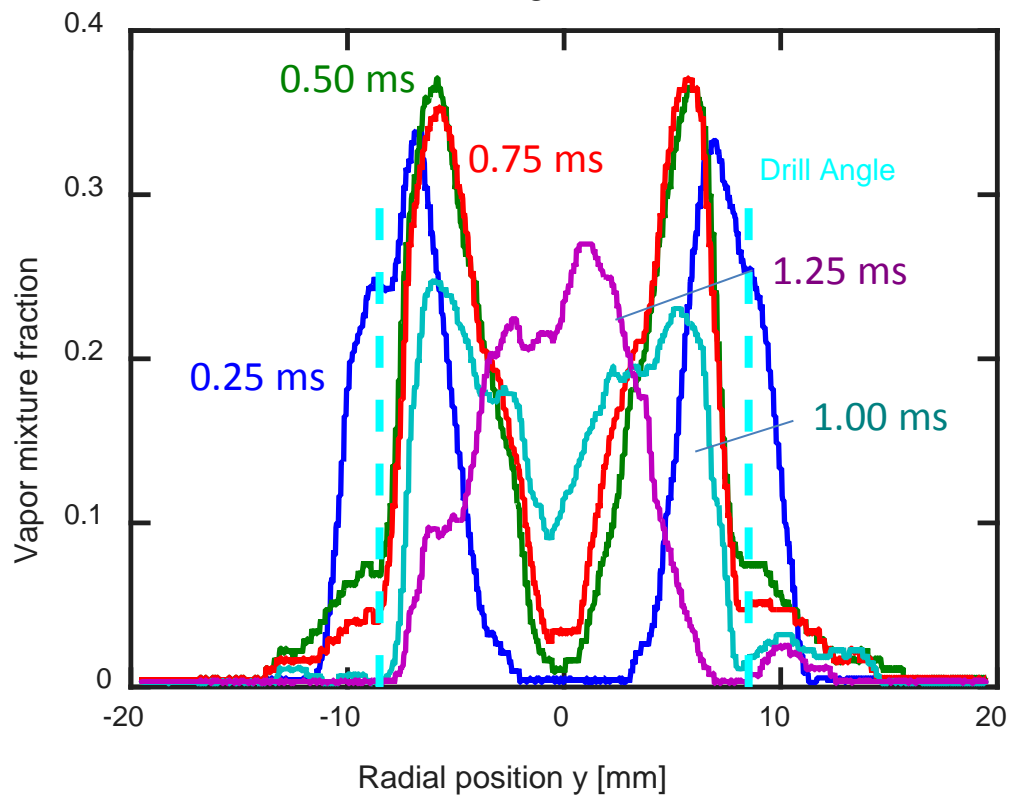


Delphi patternation at 1 atm:
 $z = 50 \text{ mm}$, $R_{\text{plume}} = 33.2 \text{ mm}$
 plume vector angle = 33°



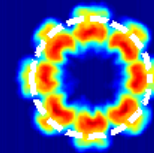
Ensemble average of 20 injections

$Z = 10$ mm

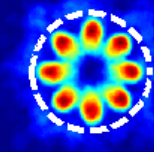


$Z = 10$ mm

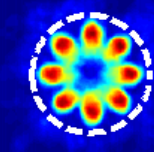
0.25 ms



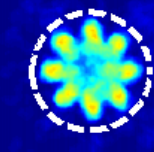
0.50 ms



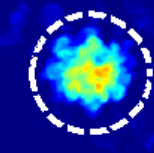
0.75 ms



1.00 ms

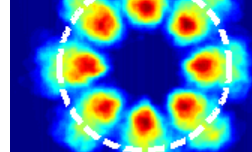


1.25 ms

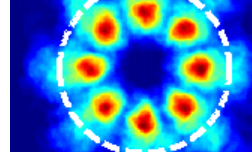


$Z = 15$ mm

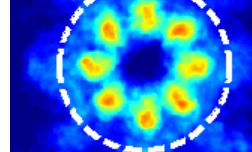
0.50 ms



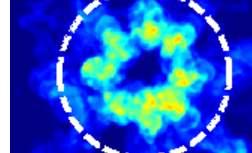
0.75 ms



1.00 ms

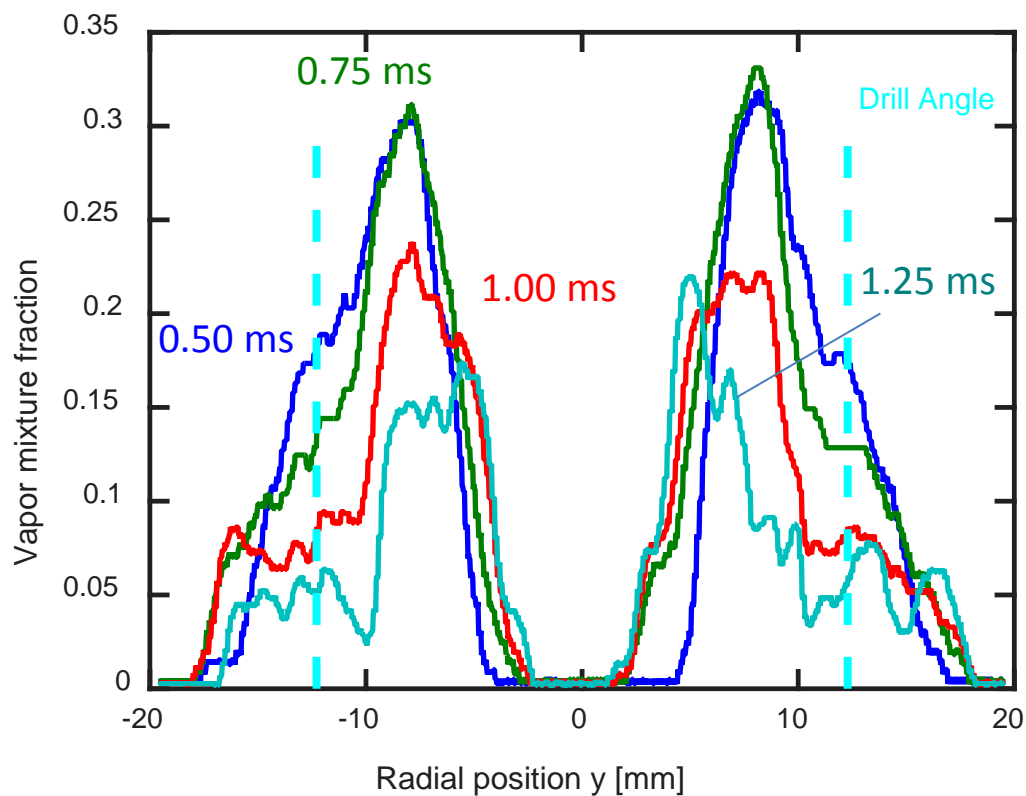


1.25 ms

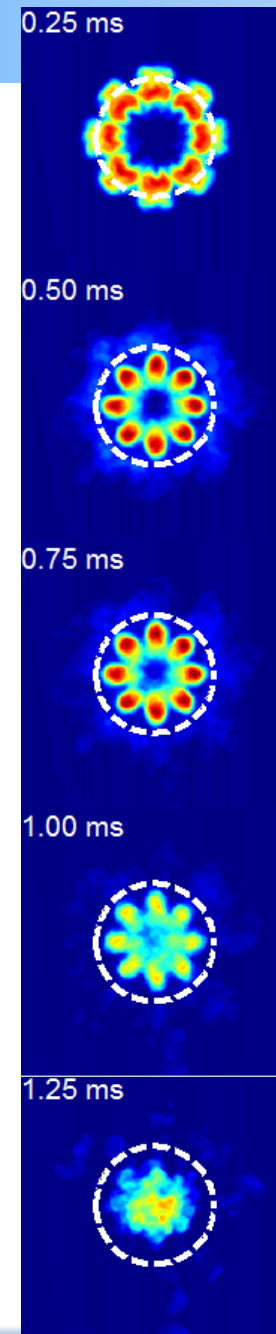




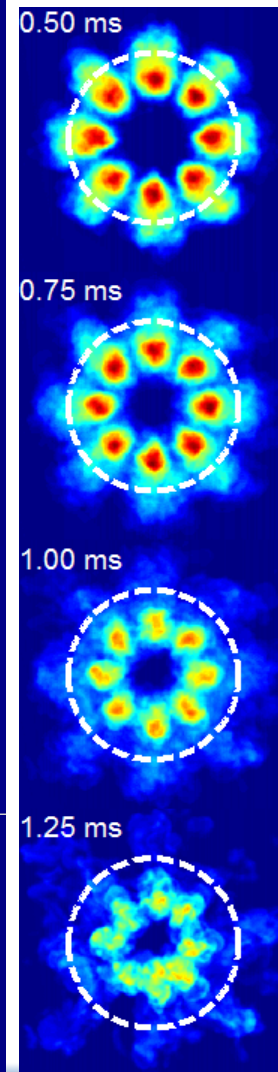
Ensemble average of 20 injections
 $Z = 15$ mm



$Z = 10$ mm



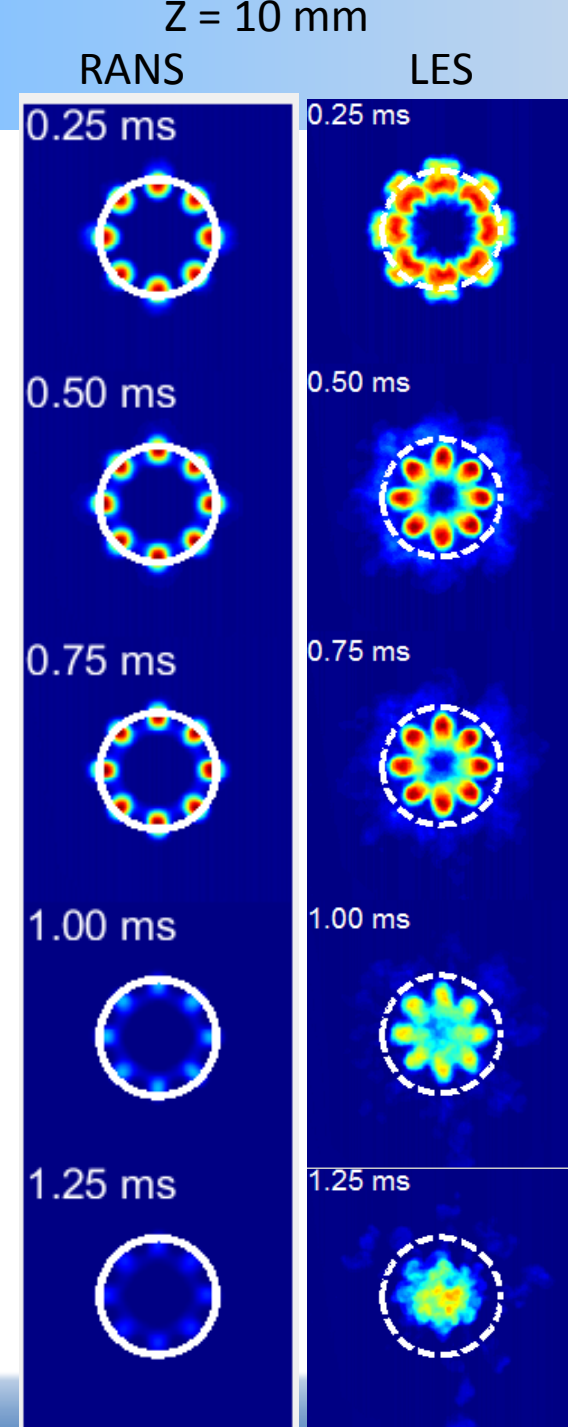
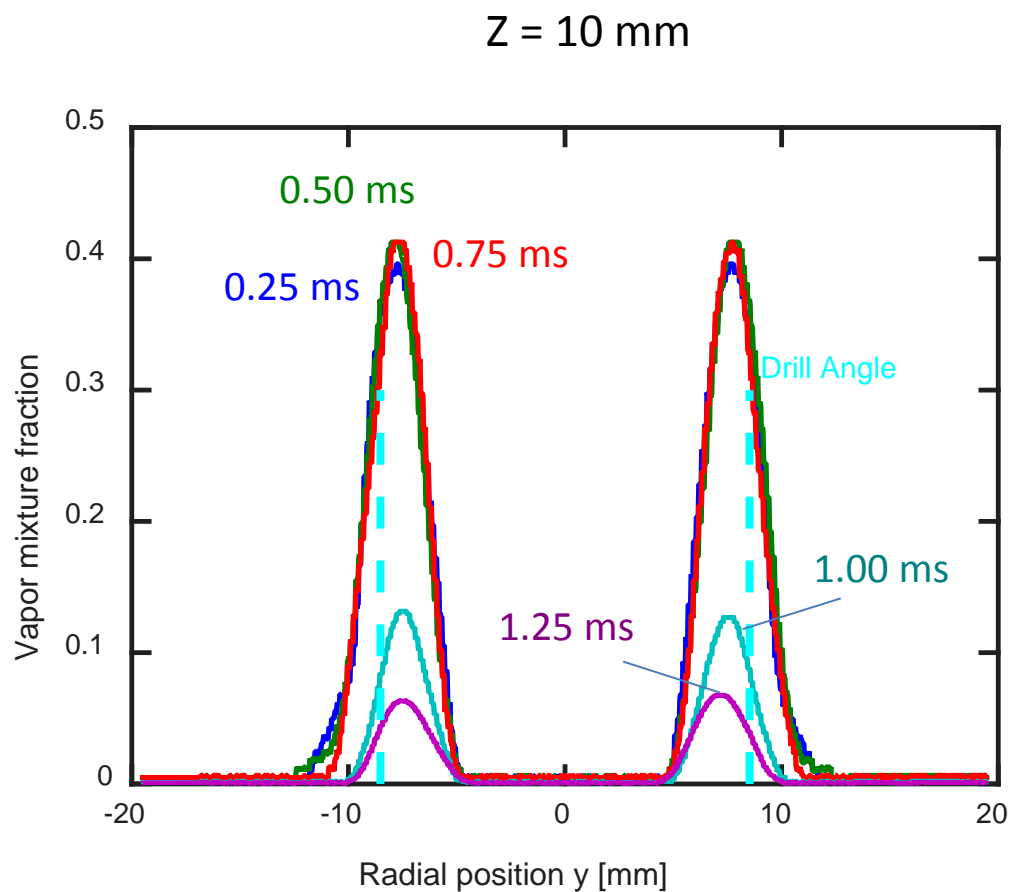
$Z = 15$ mm



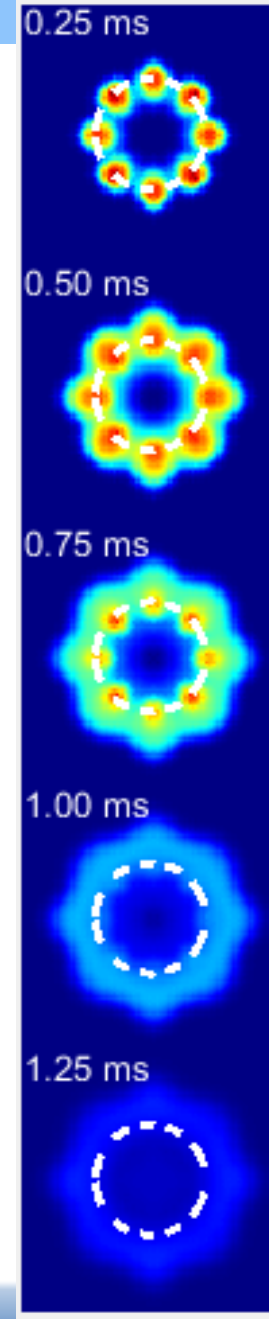
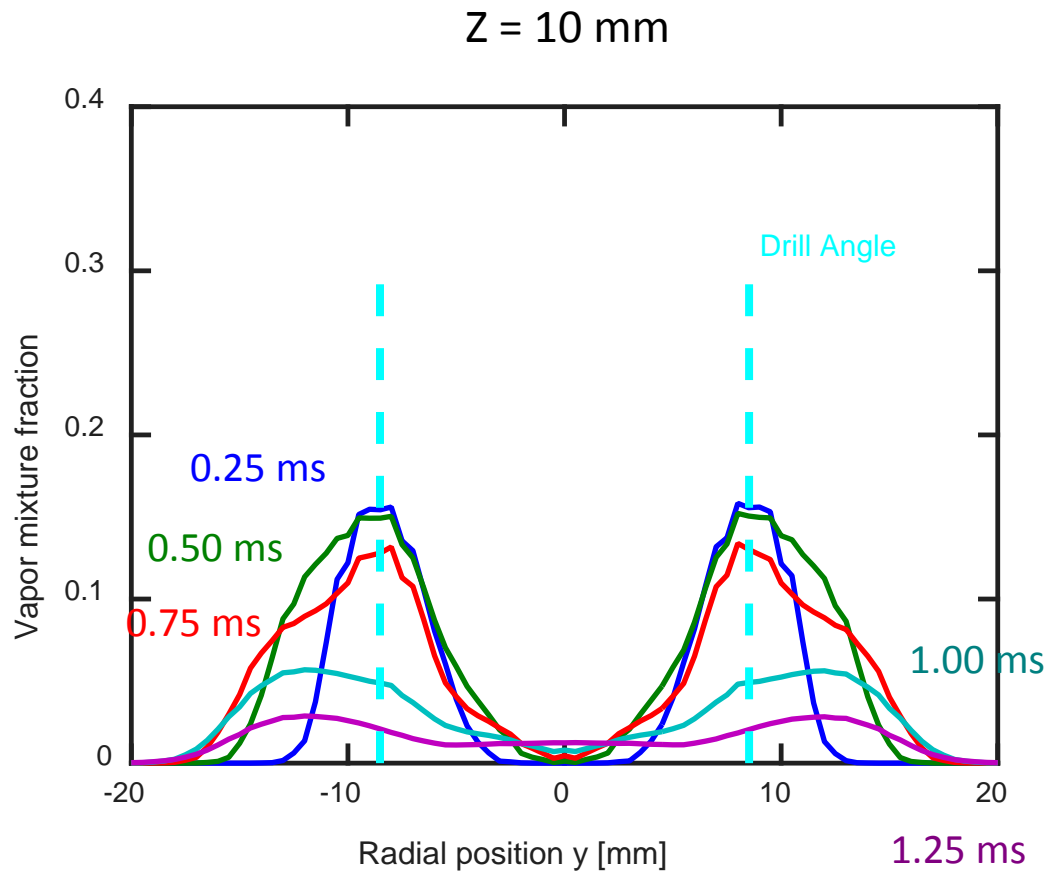


Argonne RANS vs LES simulations

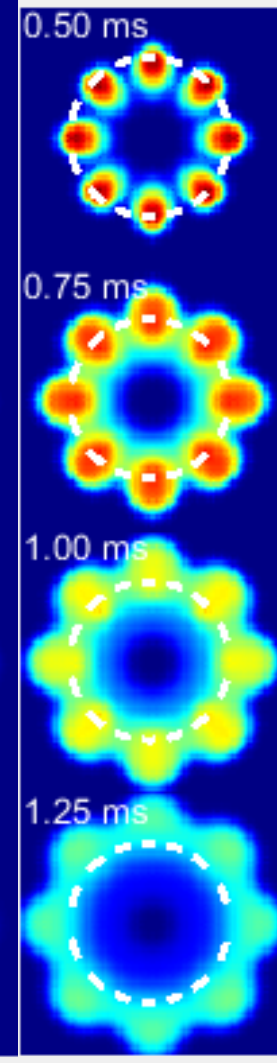
Z = 10 mm



Z = 10 mm



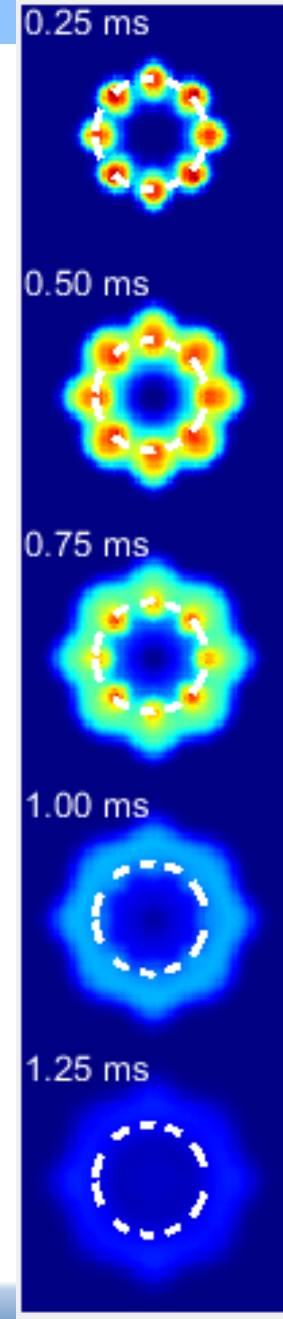
Z = 15 mm



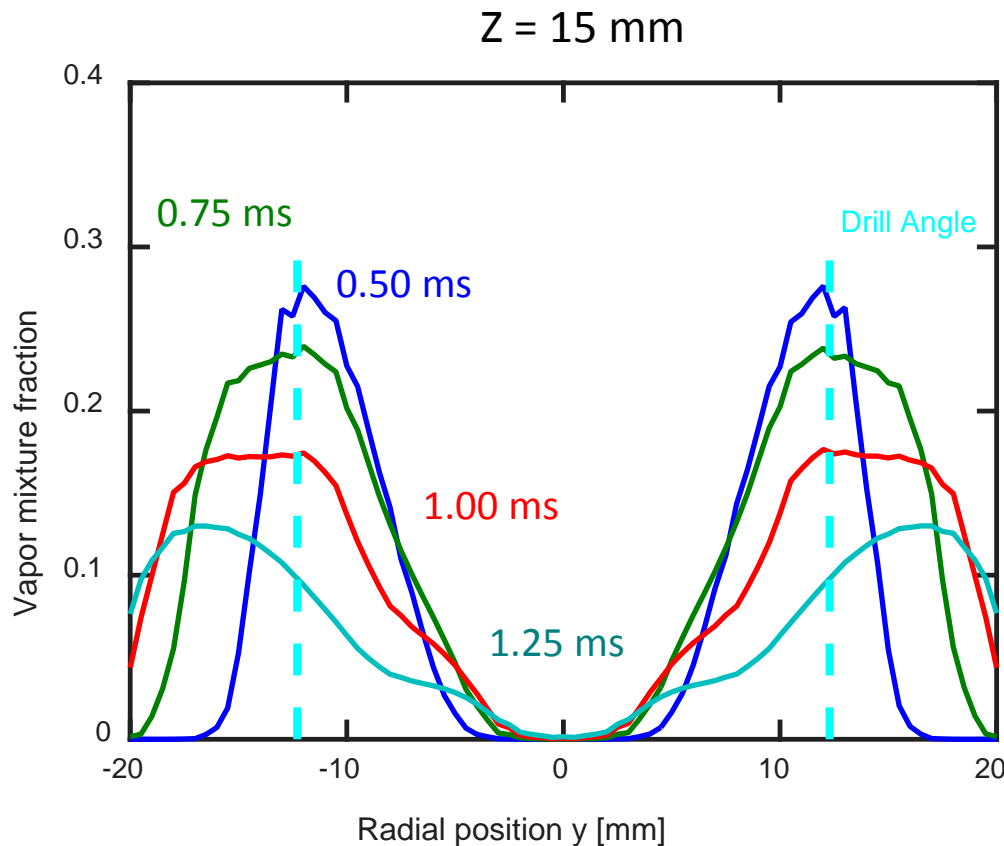
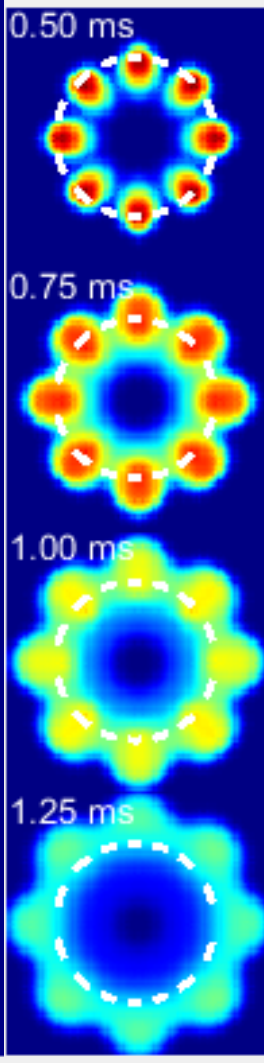


Polimi RANS simulations

Z = 10 mm

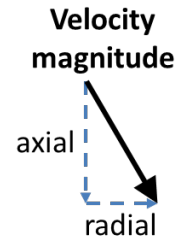
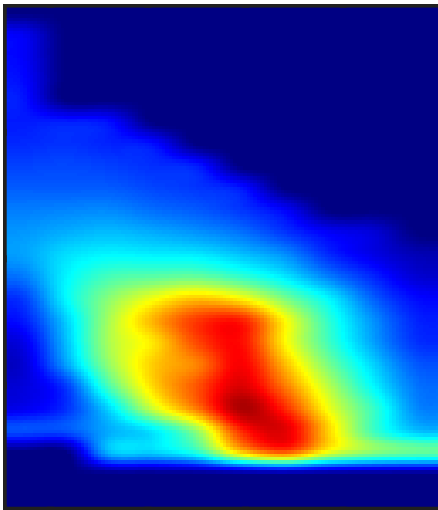


Z = 15 mm

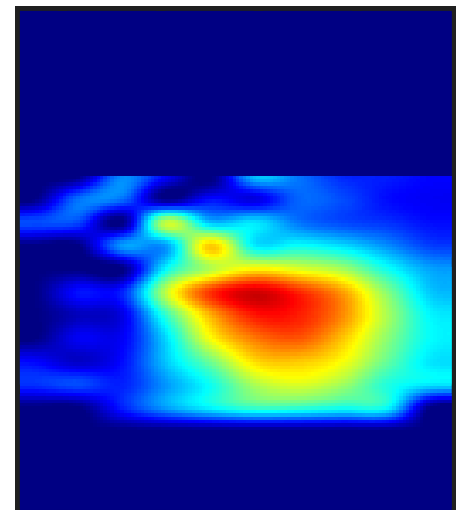


- Plume movement to injector axis does not happen in either Argonne or PoliMi RANS simulations
 - Different than experiment or LES simulations

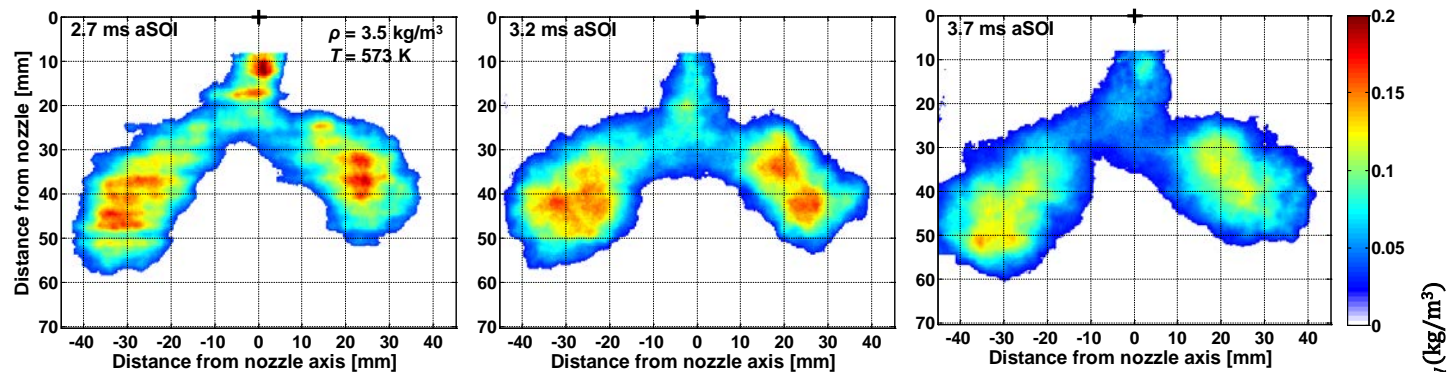
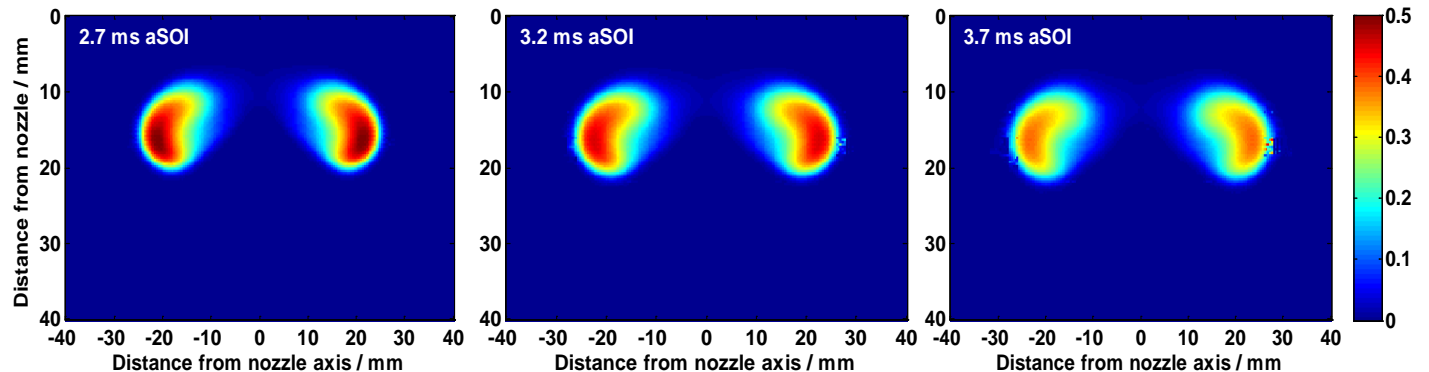
experimental
GM



RANS model
PoliMi



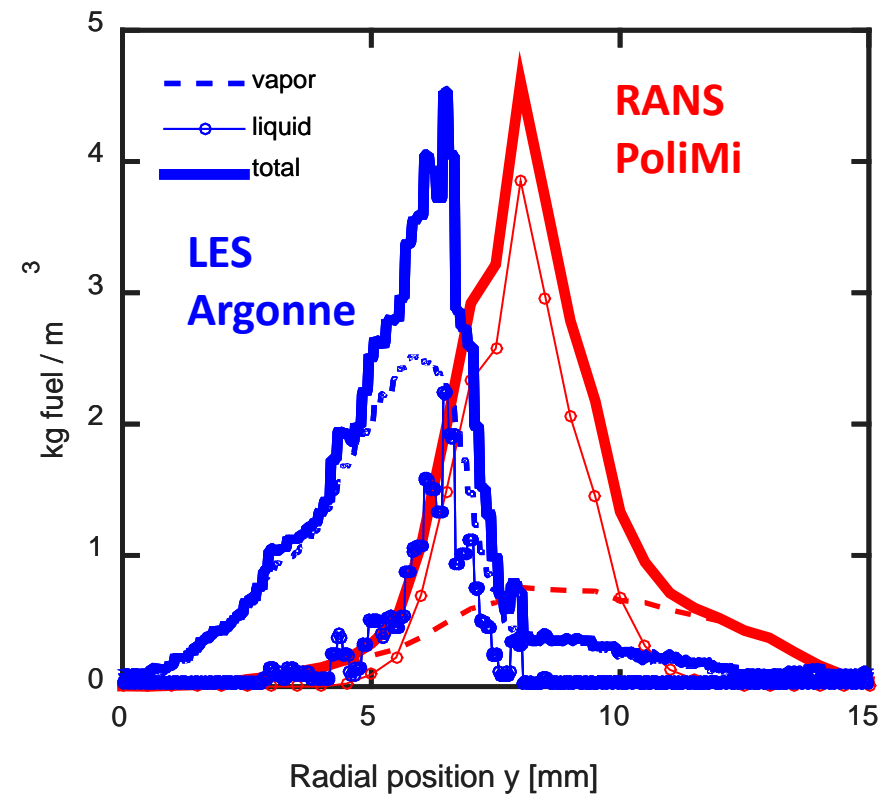
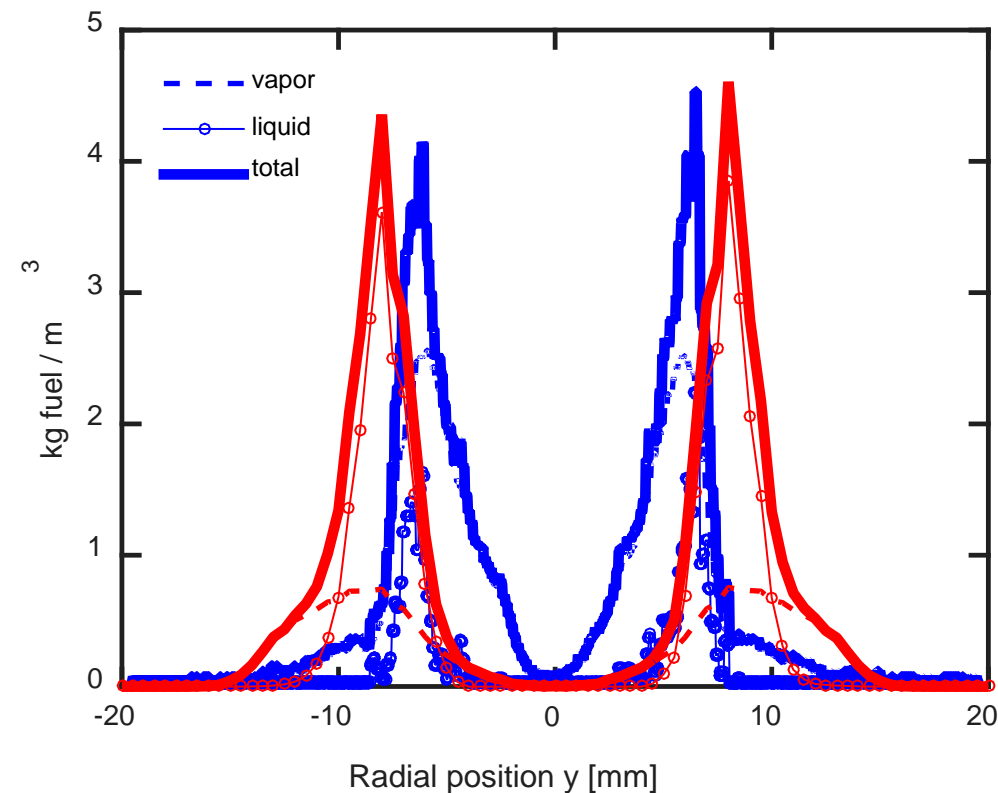
$Z = 15 \text{ mm}$

Experimental
Spray GRANS
Spray G

- Spray penetration RANS < LIF measurements
- Spray width/length RANS < LIF measurements

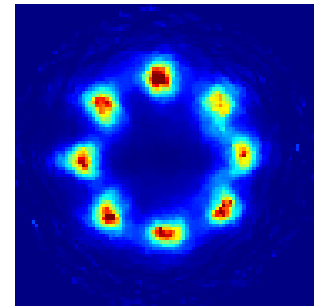
- Substantially more vaporization in the LES simulation

Z = 10 mm, 0.5 ms ASI



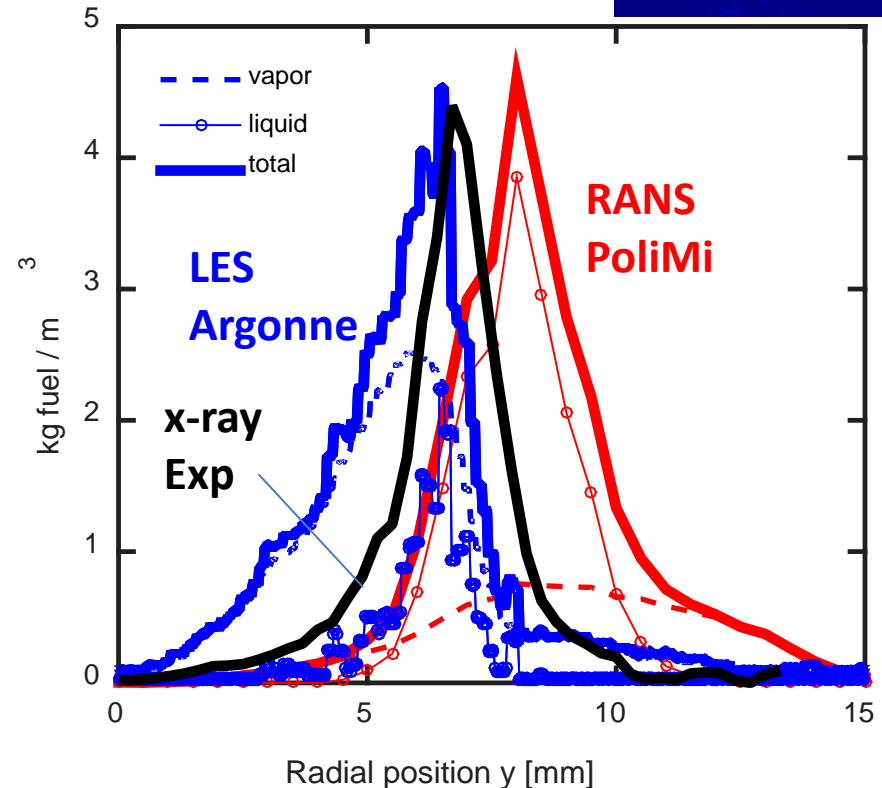
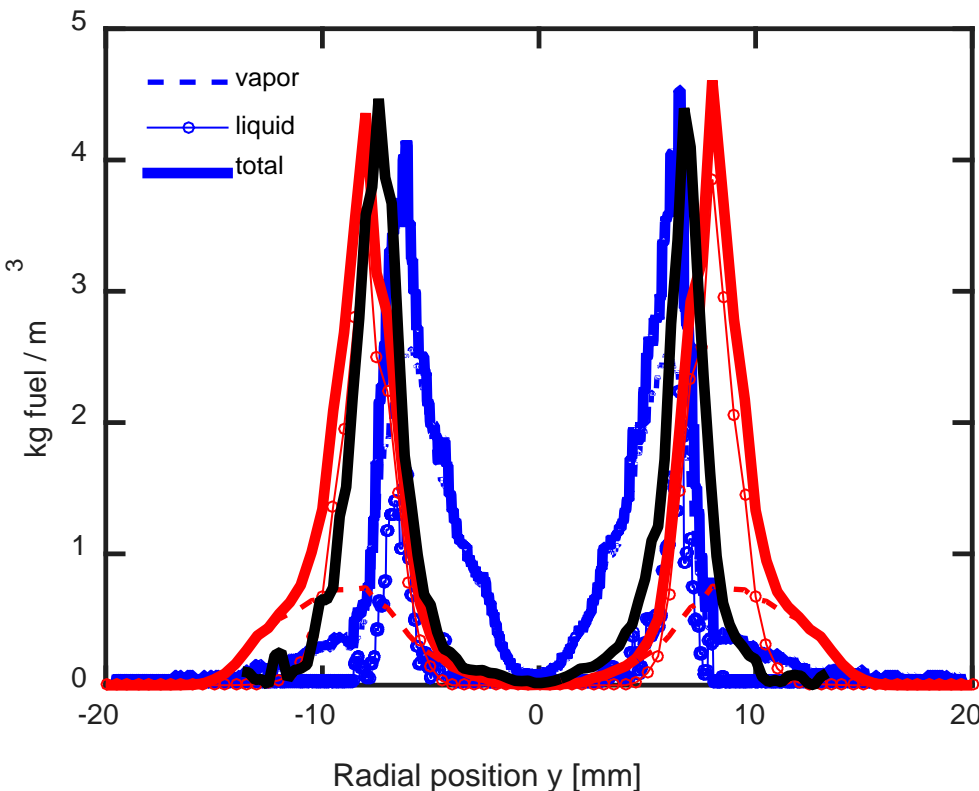
Is vaporization rate a factor for plume shift?

Argonne
radiography
0.5 ms ASI

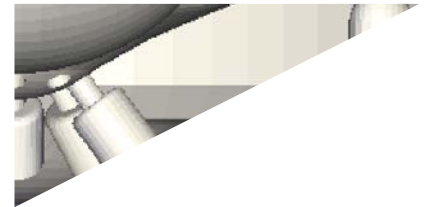


- Non-vaporizing radiography experiments show less plume dispersion
- Shift of plume center to injector axis more pronounced for LES simulation

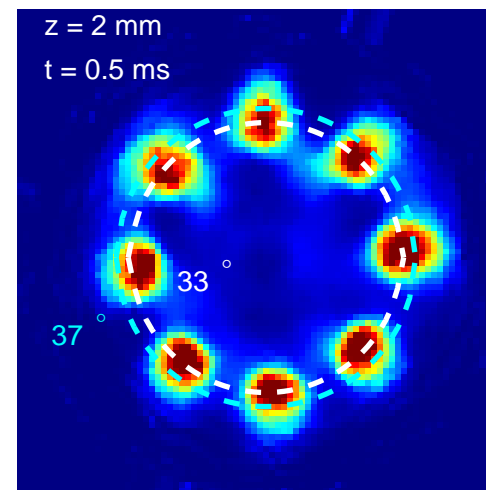
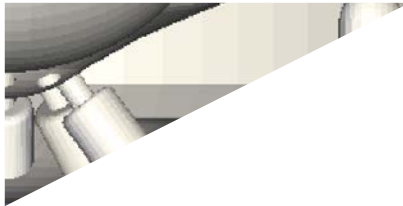
$Z = 10 \text{ mm}$, 0.5 ms ASI



- Plume center moves toward injector axis during and after injection. What is the cause?
 - Experimental evidence is convincing
 - PDI velocity measurements at $z = 15$ mm indicate plume vector (or inclination angle) is $< 31^\circ$
 - Delphi patternation data show plume vector angle = 33°
 - Argonne radiography shows plume vector angle from 34° to 32° by $z = 10$ mm
 - Always less than drill angle (37°)
 - Quantitative data describing the velocity between plumes, time resolved during injection, would be useful.
 - Measurements showing the plume direction right at the exit of the nozzle are needed.
 - Closest radiography at $z = 2$ mm
 - Help from internal flow modeling is needed



- We need recommended practices for “affordable” CFD simulations
 - Lagrangian spray modeler with somewhat coarse mesh
 - What is best practice for specifying plume vector as a function of time?
 - For only “steady” period of injection: Drill angle is too large, downstream measurements too small (plume interaction driven)
 - What grid size is simply too coarse to pick up plume interaction?
 - Method for treating counterbore and tip geometry protrusion?



Argonne
radiography
 $\text{kg fuel} / \text{m}^3$

- Experiments
 - Have not provided quantitative vapor/liquid in “mixed” regions
 - Radiography limited to non-vaporizing conditions
 - Droplet velocity measurements made, but gas/droplet velocity over larger ROI is needed
- CFD (Lagrangian)
 - Generally, RANS simulations too coarse to resolve inter-plume fluid mechanics
 - Do not consider that plume direction (moving out of the nozzle) changes with time
 - consider that small- and multi-injection cycles are common
 - Appear to have widely varying vaporization predictions