



Engine Combustion Network

ECN 6 - Diesel Combustion Committee

UNIFIED GUIDELINES FOR:

- Topic 4. Reactions that lead to ignition [Evatt Hawkes \(UNSW\)](#)
- Topic 5. Combustion (lift-off, flame structure) [B Somers \(TUE\)](#), [JM Garcia-Oliver \(CMT\)](#)
- Topic 6. Emissions (soot, NO_x, UHC) [Scott Skeen \(Sandia\)](#)
- Topic 7. Spray B (diesel) in engines [Louis-Marie Malbec \(IFPEN\)](#)
- Topic 8. Fuel effects on compression ignition [Yuanjiang Pei \(Aramco\)](#)

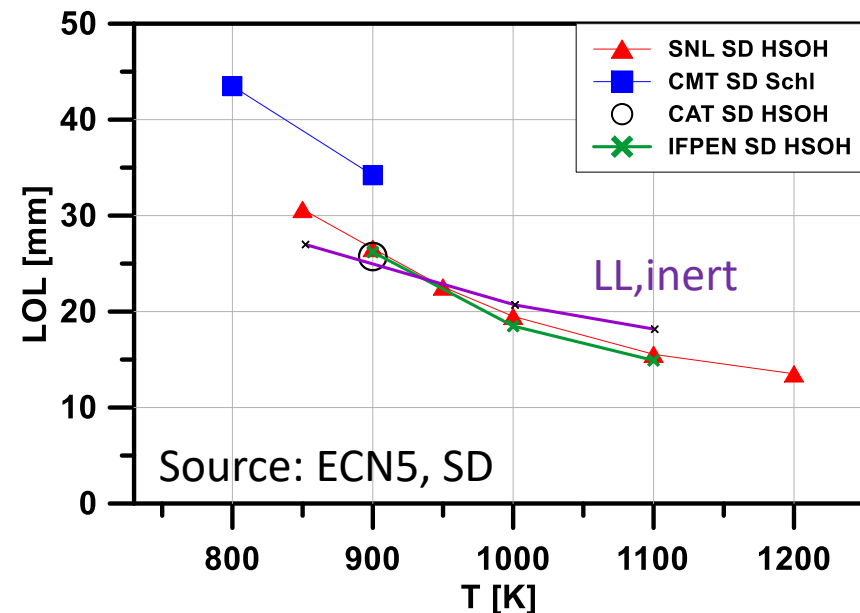
May 3rd, 2018

- INTRODUCTION
- TOPICS
 1. IGNITION SEQUENCE
 2. FLAME STRUCTURE
 3. HEAT RELEASE RATE
 4. MULTIPLE INJECTION
 5. SOOT
 6. ENGINES
 7. FUELS
- OPERATING CONDITIONS
- SOME WORDS ON CFD
- DEADLINES & PARTICIPANTS
- APPENDIX: SUBMISSION INSTRUCTIONS



- General guidelines to include different ECN groups working on **Diesel combustion** topics
 - For each topic:
motivation – objectives – requested info
- Main drivers for ECN6 compared to previous ECN
 - Merging SA/SC/SD analyses
 - How large are differences among modellers regarding HRR?
 - What improvements does LES bring?
 - Consideration of multiple injections under reacting conditions, a highly transient problem

- SA: Large experimental and modelling database
- SC/SD: Experimental database is already available
 - No CFD modelling up to now
 - Interesting test cases due to interaction between evaporation and combustion
- For ECN6, a comparison of SA/SC/SD should be carried out
 - Nozzle diameter could be considered just another parametric variable
 - To keep a reasonable effort, only reference conditions are requested for SC/SD nozzles

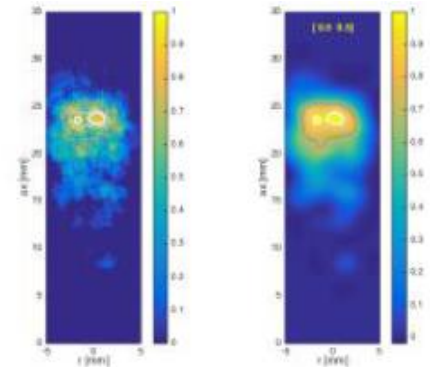


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- **MOTIVATION**

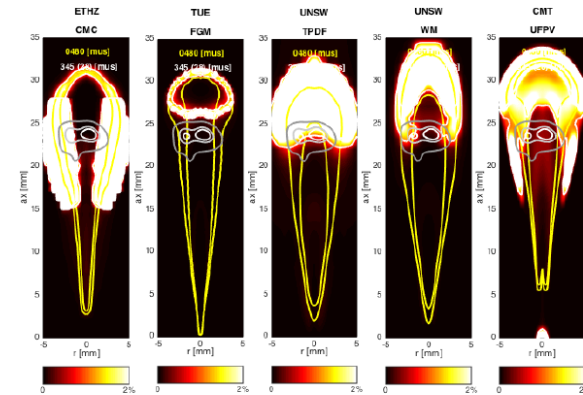
- ECN5 has shown important differences among models on ignition location and sequence
- Experiments have shown higher probability for ignition in radial periphery. Head-on view shows a “ring” of ignition for Spray A (C/D as well). What does model show from “head-on” view?
- What is the impact of slight changes in cone angle on early penetration, mixing and therefore ignition?
- Does LES bring new info into the game?

ECN4-Ignition Probability



ECN4-Ignition CFD

Base(900K, 15%)
Same time instances



- **OBJECTIVE**

- Evaluation of ignition location & timing
 - How similar/different is this picture among nozzles?
 - Sensitivity to cone angle in the spray modelling
- A comparison of Z field before ignition will be carried out (topic 3)

- **REQUESTED INFO (detailed instructions in appendices)**

- Experiments
 - Mixture fraction, location and timing at ignition
 - IFPEN experiments for ECN5 available
 - Other groups doing similar things are welcome
 - Review of ECN experimental database to derive line-of-sight ignition locations
 - Detailed penetration measurements are needed, especially at early timings.
- Model
 - Ignition delay timing
 - Field variables: Time- and spatially resolved species from SOI (Start of Injection) to slightly after SOC (start of combustion)
 - Small variation of cone angle to check sensitivity of ignition to mixing field

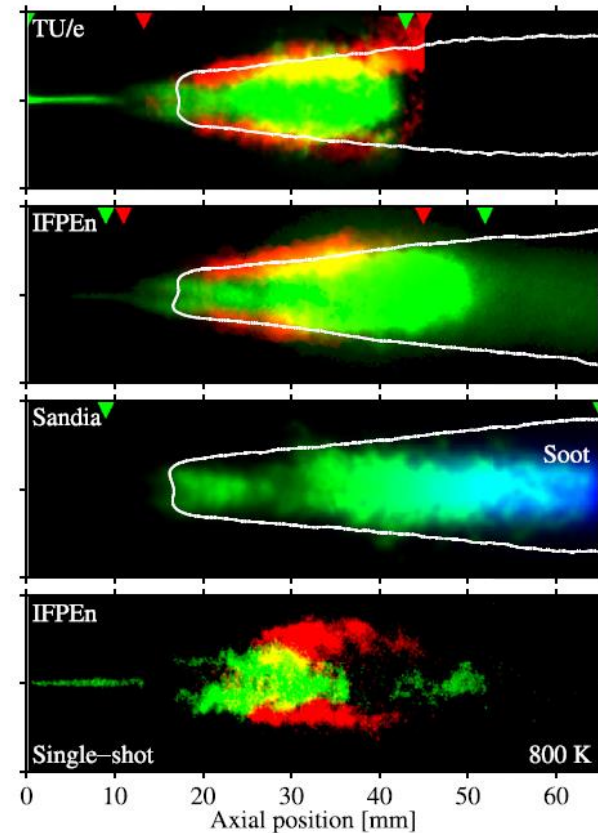
- **MOTIVATION**

- SA flame structure analysed in detail in past ECN workshops
- Can models capture
 - the transition from SA to SD?
 - Differences between SC/SD?
- Does LES bring in new info?

- **OBJECTIVE**

- Evaluation of accuracy of models for flame structure definition in both small and large nozzles
 - LOL
 - Detailed spatial structure: CH₂O, OH fields
 - Z field under reacting conditions

SA

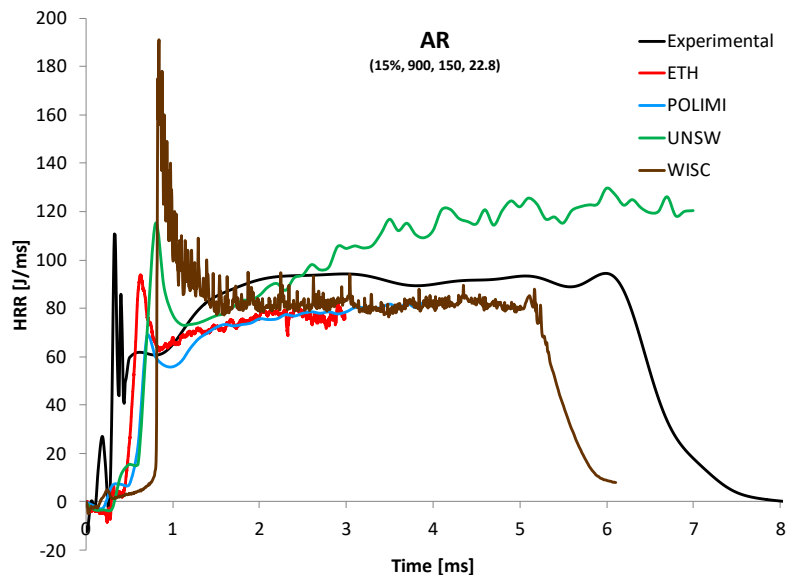


N. Maes et al. / Combustion and Flame 174 (2016) 138–151

- **REQUESTED INFO (detailed instructions in appendices)**
 - Experiments
 - SA/SC/SD: LOL from OH*
 - Time-resolved high-speed OH*, comparison among ECN facilities in progress
 - SC/SD: Detailed spatial structure: CH₂O, OH fields
 - Modelling
 - Time-resolved LOL
 - Field variables: Time- and spatially resolved species from start of combustion (SOC) until end of injection (EOI)

- MOTIVATION**

- How accurate/similar are our modelling approaches in terms of heat release rate (HRR) predictions
 - Important for industry
- Introduced at ECN3, but never considered again
 - Have we improved this picture?



Source: ECN3, Combustion indicators

- **OBJECTIVE**
 - Quantification of HRR in SA/SD conditions
 - Comparison among vessels
 - Evaluate differences among models in chemical HRR
 - Coordination with Topic 7 (engines)
- **REQUESTED INFO (detailed instructions in separate doc)**
 - Experiments
 - Measurements of pressure in constant-volume cells.
 - Modelling
 - Time-resolved chemical HRR (and Pressure, if a constant-volume vessel is simulated)
 - A coordinated routine has been prepared and will be used to derive HRR from pressure

- **MOTIVATION**
 - Usual injection strategy in Diesel engines
 - Highly transient compared to single-pulse cases used until now
 - Major role of combustion model (mech + TCI) because of the interaction between pulses

- **OBJECTIVE**
 - Evaluation of predictability of current models with a split injection strategy
 - Pilot 0.5 ms , dwell 0.3/0.5/0.7 ms, main 0.5 ms
 - Coordination with Topic 3 (spray mixing)

- **REQUESTED INFO** (detailed instructions in appendices)
 - Experiments
 - Detailed penetration (schlieren)
 - LOL/ID as from OH*
 - Simulations
 - Time-resolved LOL
 - Field variables: Time- and spatially resolved species from start of combustion (SOC) until end of second injection (EOI2)

- **MOTIVATION**
 - Expanding on work from ECN5 with O₂/N₂ vs EGR (pre-burn) conditions for Spray A
- **REQUESTED INFO (detailed instructions in appendices)**
 - Time-resolved field variables (consistent with Ignition/Flame structure submission)
 - Everyone must use the SAME chemical mechanism for their primary submission of the Standard Spray A condition and the O₂/N₂ condition.

- **MOTIVATION**

- Progress towards predictive models in engine environment.
- Understand and model the effects of confinement, aerodynamics, and real engine operation (Diesel fuel, multi-hole nozzles, multi injections) on mixing and combustion processes.

- **OBJECTIVE**

- Study the effect of geometry and flow on mixing /combustion processes

- **REQUESTED INFO** (detailed instructions in appendices)

- Basic metrics of the spray in engine: liquid length, vapor penetration, lift-off length, ignition delay.
- Additional: same data in free jet configuration.
- CFD: Time resolved fields variables

- **MOTIVATION**

- Understand the fuel effects on spray at CI conditions as an additional dimension for model development and validation

- **OBJECTIVE**

- Summarize the available experimental and computational fuel effect studies for future directions

- **REQUESTED INFO**

- For both experiments and simulations for two or more fuels (different physical or/and chemical properties) at one of the following conditions - Spray A, C or D

- **REQUESTED INFO**

- Fuels used, with physical and chemical (e.g., cetane or RON number) properties
- In nozzle flow:
 - Needle lift for different fuels
 - Cavitation comparison (mainly for simulations)
- Non-reacting spray at different ambient T conditions:
 - liquid length/vapor penetration length vs. time
 - Mixing field (fuel mass fraction) at different timing if available
- Reacting spray at different ambient T conditions:
 - Ignition delay and lift-off length
 - OH* contour
 - Soot mass vs. Time or other available data
- Please use the ECN definitions, if different, please specify

• NOMINAL TEST MATRIX

PARAMETRIC VARIABLE	ACRONYM	O2 [%]	Ta [K]	Dens [kg/m ³]	Pinj [MPa]	Inj Duration
Standard Spray A	Ai	0	900	22.8	150	LONG
	Ar	15	900	22.8	150	LONG
Temperature	T2	15	800	22.8	150	LONG
Injection pressure	I1	15	900	22.8	100	LONG
	I2	15	900	22.8	50	LONG
Multiple injection	MAI/MAI2	0	900	22.8	150	0.5 ms pulse - 0.5/0.7 ms dwell - 0.5 ms pulse
	MAR/MAR2	15	900	22.8	150	0.5 ms pulse - 0.5/0.7 ms dwell - 0.5 ms pulse

– Ambient composition:

- Non-EGR: X_{O2}=15%, X_{N2}=85%
- EGR(pre-burn):
X_{O2}= 15%, X_{N2}=75.15%, X_{CO2}=6.23%, X_{H2O}= 3.62%

- **SPRAY IN VESSELS**

- Long injection (except for multiple injection): 5 ms

- Priorities for each topic

- IGNITION SEQUENCE / HRR / Soot

- Nozzle: SA/SC/SD

- Conditions: SA (EGR/non-EGR) + T2 (800K)

- FLAME STRUCTURE / Soot

- Nozzle: SA/SD

- Conditions: SA (EGR/non-EGR)

- MULTIPLE INJECTIONS

- Nozzle: SA

- Conditions: MAI/MAR

- **SPRAY IN ENGINES**
 - Injection duration: 1.5ms
 - Spray B
 - Variations
 - Temperature: 800K and 1000K
 - Injection pressure: 50 and 150MPa
 - Density: 15.2kg/m³
 - %O₂: 13% and 21%
 - Swirl or engine speed
 - Notes:
 - These conditions are for TDC
 - SOI: near TDC
 - Geometry: engine (+free jet if possible)
 - If possible, variable liquid angle



- **TCI model**
 - Reference well-mixed
 - Each participant can additionally submit calculations with a particular TCI closure

- **Chemical mechanism**
 - Reference (ALL groups should use):
T. Yao, Y. Pei, B.-J. Zhong, S. Som, T. Lu, et al., A compact skeletal mechanism for n-dodecane with optimized semi-global low-temperature chemistry for diesel engine simulations, Fuel 191 (2017) 339-349.
 - Additional submissions with a different chemical mechanism are welcome, but not required.



- **RANS vs LES**
 - Reference RANS
 - LES submissions are welcome

- **Soot models**
 - Reference (ALL groups should use):
K.M. Leung, R.P. Lindstedt, W.P. Jones, A simplified reaction mechanism for soot formation in nonpremixed flames, Combustion and Flame (1991) 87(3–4) 289-305

• SUBMISSION OF MODELLING FIELD VARIABLES

– Field variables to be reported:

Data	ACRONYM	
Axial velocity (m/s)	U	
Radial velocity (m/s)	V	
Mixture fraction	Z	
Temperature (K)	T	
Density (kg/m ³)	RHO	
Mixture fraction variance	Zvar	OPTIONAL
Turbulence kinetic energy (m ² /s ²)	K	OPTIONAL
Turbulence kinetic energy dissipation rate (m ² /s ³)	EPS	OPTIONAL
Viscosity (molecular, kg/m/s)	VIS	OPTIONAL
Viscosity (turbulent, kg/m/s)	VIST	OPTIONAL
Scalar dissipation rate (1/s)	CHI	OPTIONAL

MORE DETAILS IN APPENDICES

Data	ACRONYM
Species mass fraction (kgi/kg)	YC12H26
	YO2
	YCO2
	YH2O
	YCO
	YCH2O
	YOH
	YCH
	YH
	YO
	YC3H3
	YHCO
	YC2H2
	Production/destruction rate of species (kgi/m3/s)
$\dot{\omega}$ YO2	
$\dot{\omega}$ YCO2	
$\dot{\omega}$ YH2O	
$\dot{\omega}$ YCO	
$\dot{\omega}$ YCH2O	
$\dot{\omega}$ YOH	
$\dot{\omega}$ YCH	
$\dot{\omega}$ YH	
$\dot{\omega}$ YO	
$\dot{\omega}$ YC3H3	
$\dot{\omega}$ YHCO	
$\dot{\omega}$ YC2H2	
Soot Volume Fraction	

• SUBMISSION OF MODELLING FIELD VARIABLES

MORE DETAILS IN APPENDICES

– Time discretization:

- From Start of Injection (SOI) until 100 μs after Start of Combustion (SOC+100) field variables should be reported with $\Delta t = 10 \mu\text{s}$
- From 100 μs after SOC until End of Injection (EOI) field variables should be reported with $\Delta t = 50 \mu\text{s}$

– Spatial resolution: Uniform Cartesian mesh

- Radial: 0 to 20mm; 0.04mm spacing (501 points)
- Axial: 0 to 100mm; 0.2mm spacing (501 points)(if using a smaller mesh, the spacing should be maintained).

- SUBMISSION DEADLINES
 - All the requested information should be submitted by Monday June 18th 2018
- PARTICIPANTS

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